

ISTITUTO  
DI TECNOLOGIE DELLA  
COMUNICAZIONE,  
DELL'INFORMAZIONE  
E DELLA  
PERCEZIONE



Scuola Superiore  
Sant'Anna

## A comparative assessment of performance of active exoskeletons for haptic feedback: tendon driven vs harmonic drive based designs

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HS' 2012  
Workshop on Hardware Evaluation

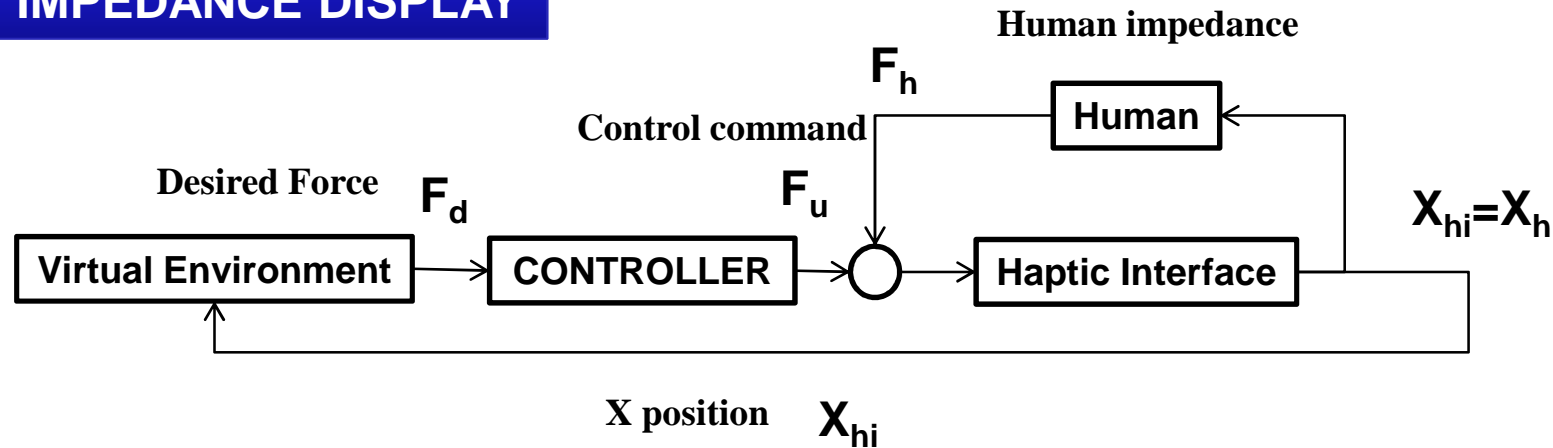
# Wearable devices development - exoskeletons

- The rationale for the design of high performance haptic interface are the satisfaction of the requirements of
  - high force fidelity
  - transparency
  - backdrivability.
- It is well known that two main basic approaches can be adopted in the design of haptic interfaces and more specifically of active exoskeletons:
  - impedance based design
  - admittance based design

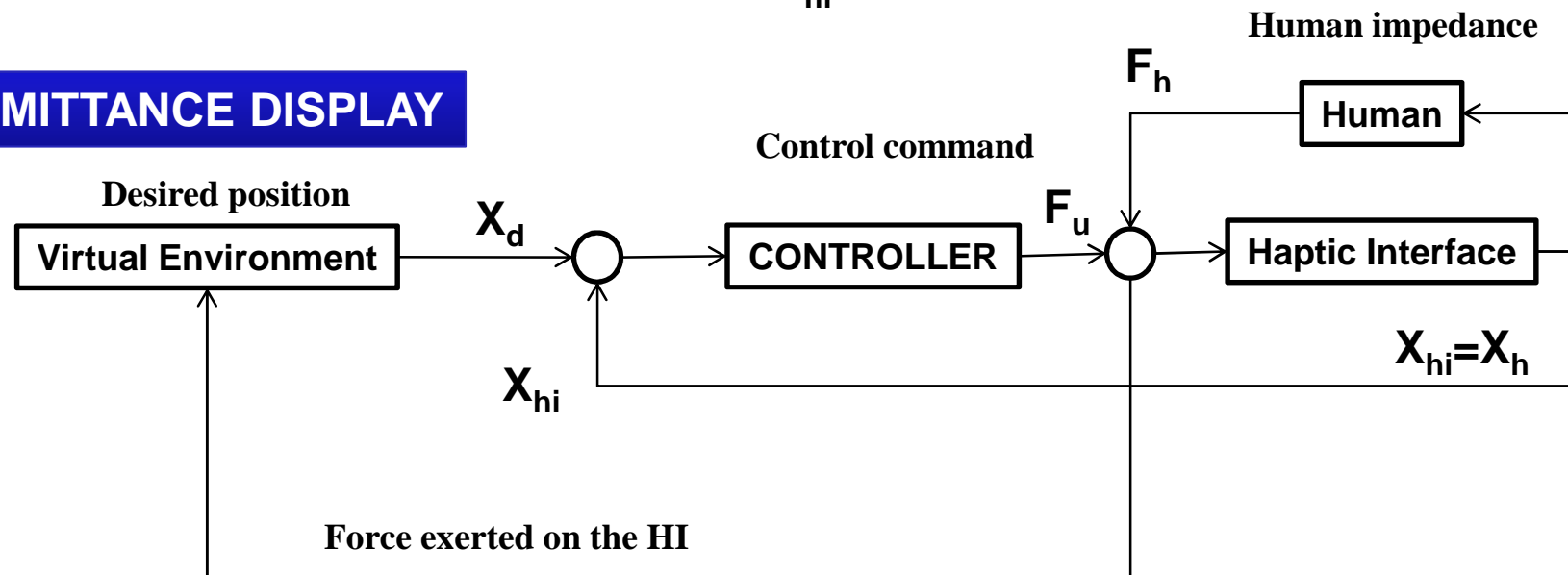


# Admittance vs. impedance display

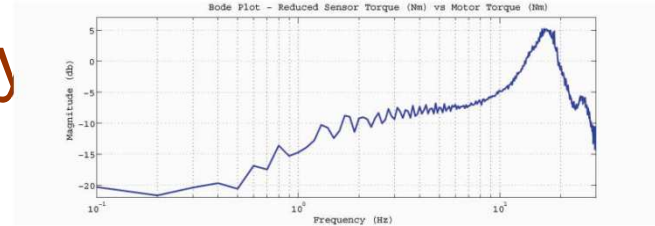
## IMPEDANCE DISPLAY



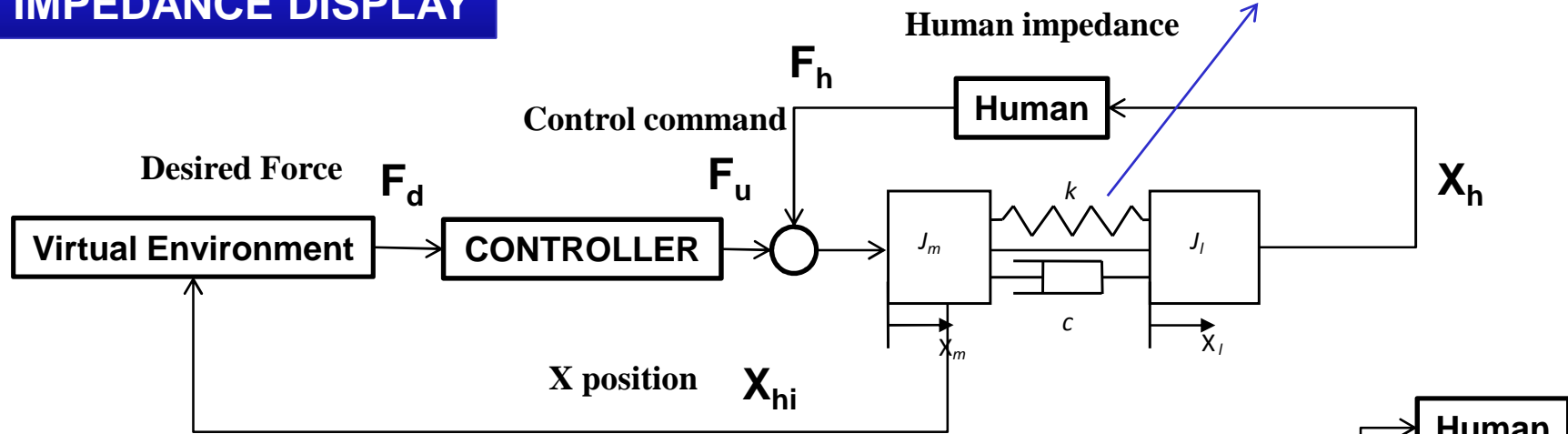
## ADMITTANCE DISPLAY



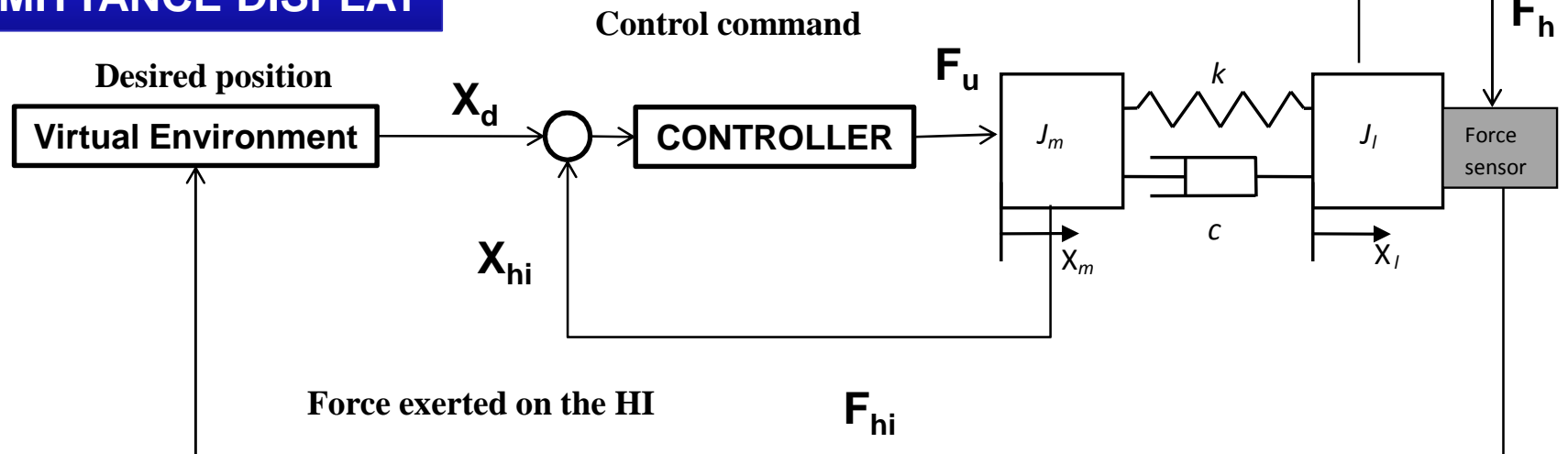
# Admittance vs. impedance display



## IMPEDANCE DISPLAY



## ADMITTANCE DISPLAY



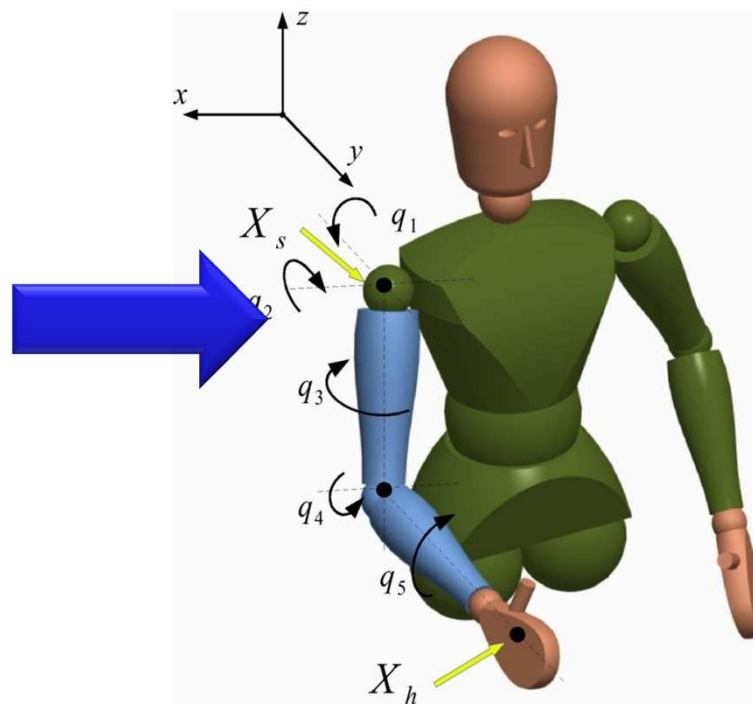
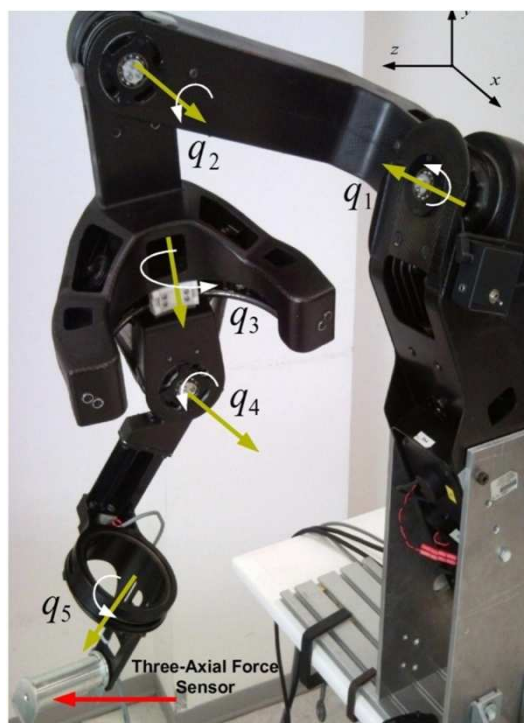
A tendon driven exoskeleton

# THE L-EXOS SYSTEM



**Antonio Frisoli, Caterina Procopio, Carmelo Chisari, Ilaria Creatini, Luca Bonfiglio, Massimo Bergamasco, Bruno Rossi, Maria Chiara Carboncini, "Positive effects of robotic exoskeleton training of upper limb reaching movements after stroke", accettato per la pubblicazione, in stampa in Journal of Neuroengineering and Rehabilitation (IF 2.638) (May 2012)**

# System overview



The basic movements of the human arm are controlled for all time, attenuating undesired excessive motions of each human arm movement, i.e., abduction/adduction, flexion/extension, internal/external rotation and pronation/ supination

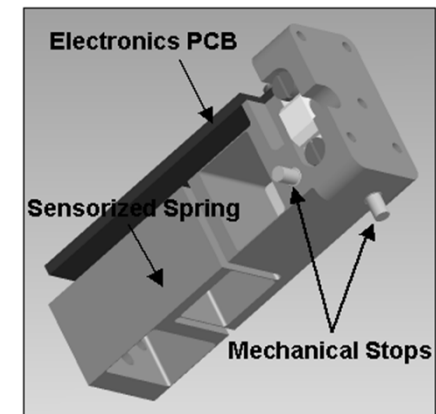
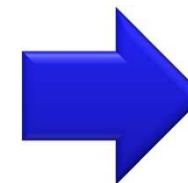
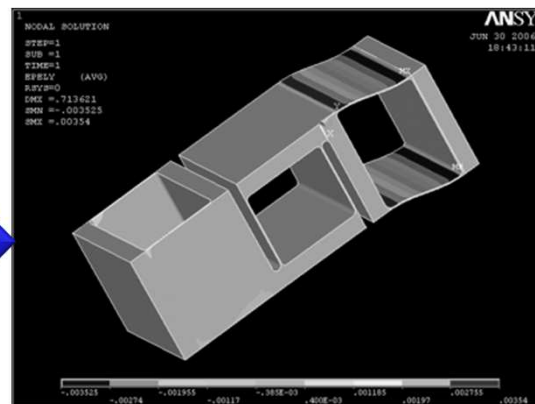
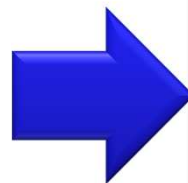
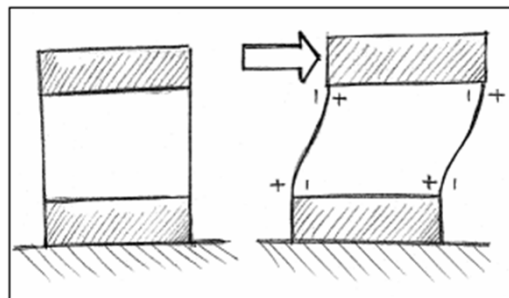
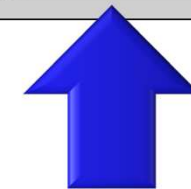
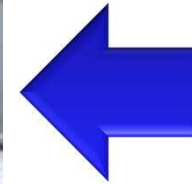
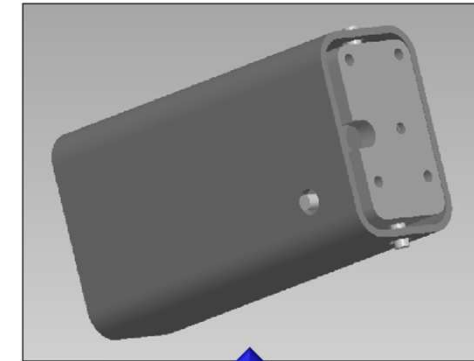
- L-exos is characterized by a serial kinematics consisting of five rotational joints.
- The first three rotational axes are incident and mutually orthogonal (two by two) in order to emulate the kinematics of a spherical joint with the same center of rotation of the human shoulder.
- Handle is sensorized with a custom design force sensor

**Frisoli, A. Montagner, L.i Borelli, F. Salsedo, M. Bergamasco, "A force-feedback exoskeleton for upper limb rehabilitation in Virtual Reality", Applied Bionics and Biomechanics 6(2), pp115-126, (2009)**

# EF forse sensing

## -Three Axial Force Sensor.

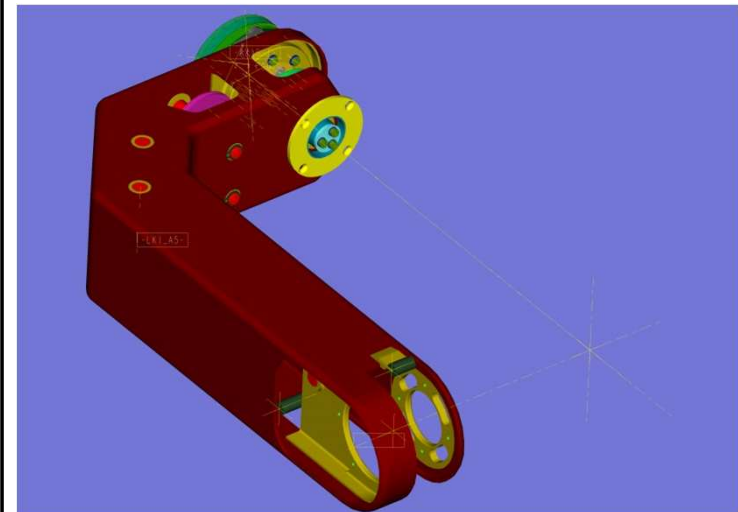
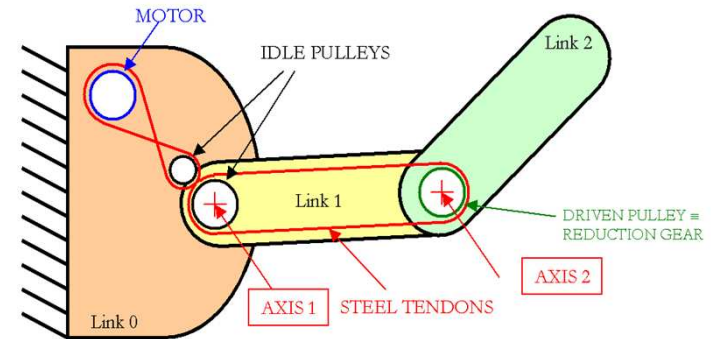
- Maximum Load: 100 N.
- Resolution 0.0500 Volts/N.
- Three axial sensing.
- Nonlinearity 3%.
- Made in PERCRO.



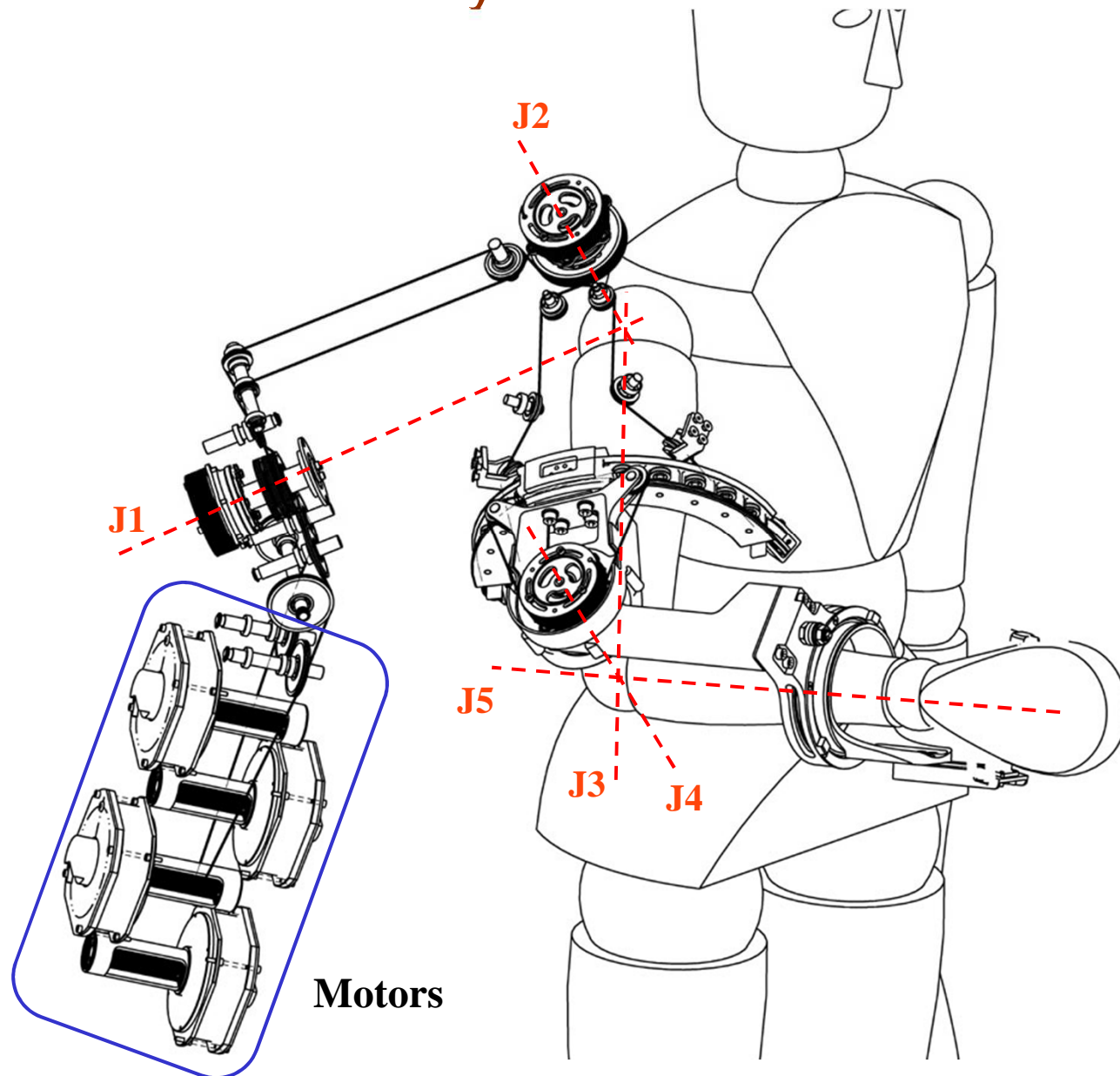


# Mechanical Constructive Features

<p><b>All the motors located on Link 0</b></p>	<p><b>Reduction of the masses of the moving parts near 40%</b></p>
<p><b>Reduction gear integrated at the joint axis</b></p>	<p><b>Increase of stiffness at the end effector</b></p>
<p><b>Structural components designed as thin-wall parts (mechanical parts integrated within the links)</b></p>	<p><b>Reduction of mass of transmission systems and of the structural parts</b></p>
<p><b>Structural components made of carbon fiber</b></p>	<p><b>Increase of stiffness at the end effector</b></p>
	<p><b>Improvement of protection for inner parts</b></p>
	<p><b>Increase of protection for user from potential harm deriving</b></p>
	<p><b>Further Improvement of lightness and stiffness</b></p>



# Exos transmission system



# Why complex tendon transmission

Improve force-feedback fidelity and transparency of use

Remote  
placement  
of motors

Tendon  
transmissions

**Benefits:**

- Reduction of mass of moving link
  - smaller motors sizes are needed
  - reduction of perceived inertia
- Opportunity of increase device ergonomy

**Benefits:**

- Reduction of backlash
  - Reduction of mass of transmission parts
  - Opportunity of increase device ergonomy  
(by integrating transmission systems inside the link)

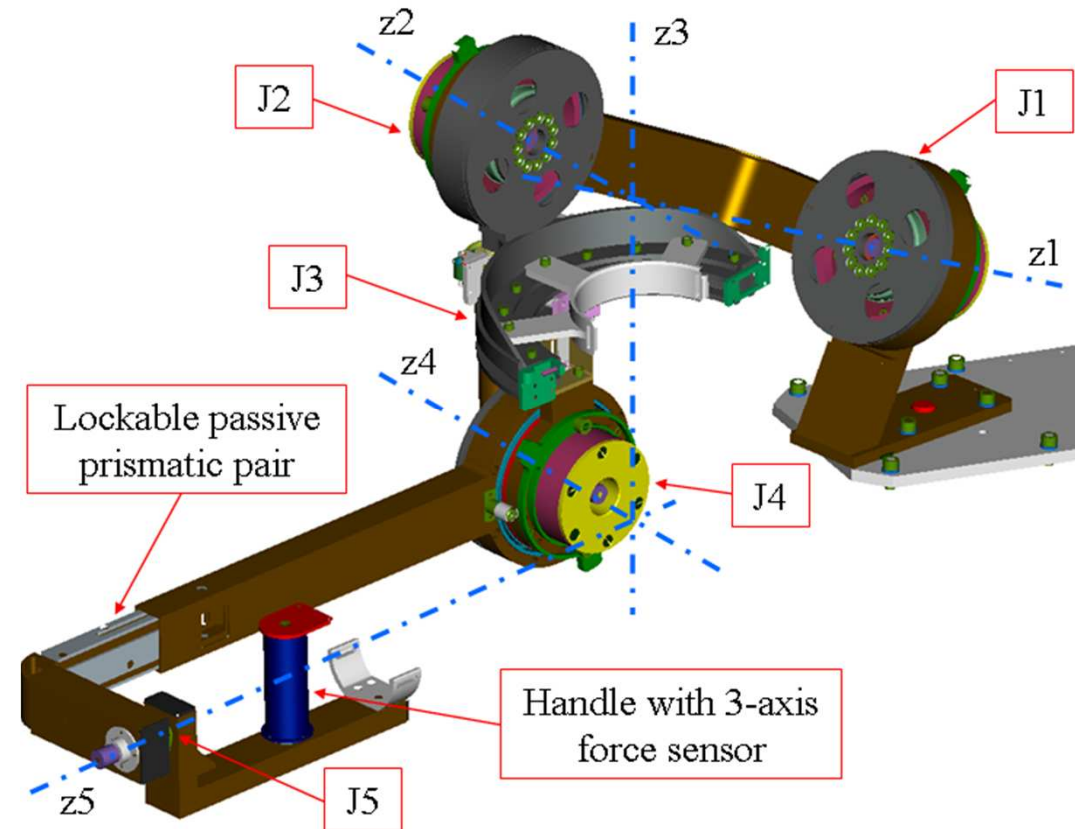
# THE REHAB-EXOS SYSTEM



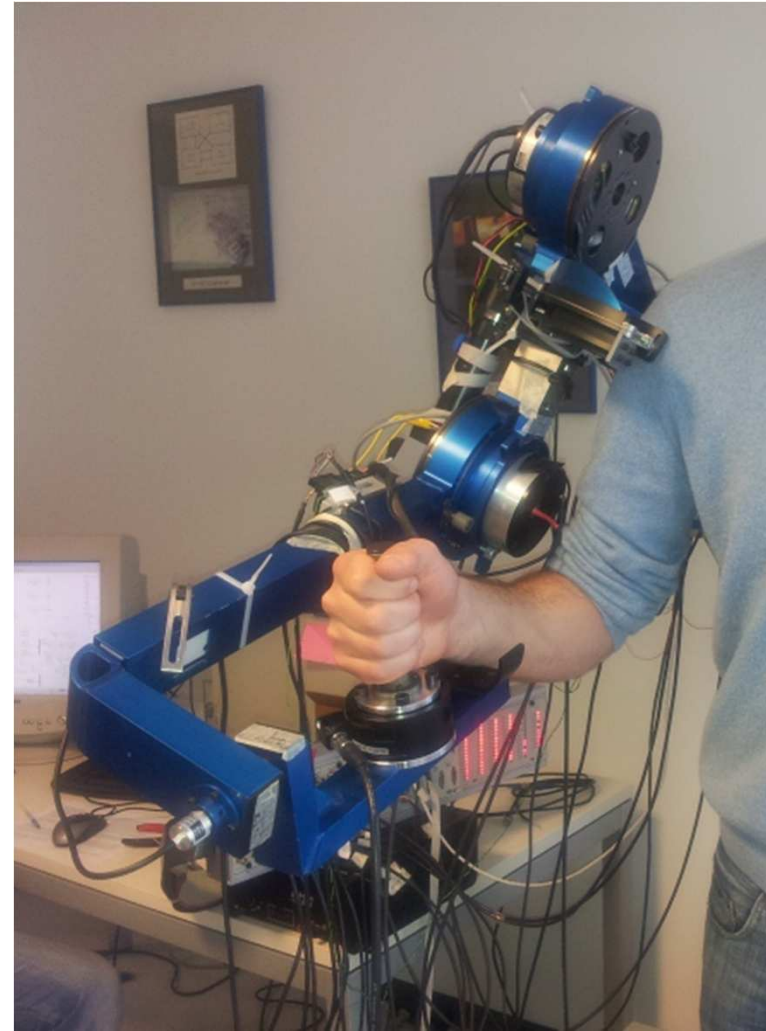
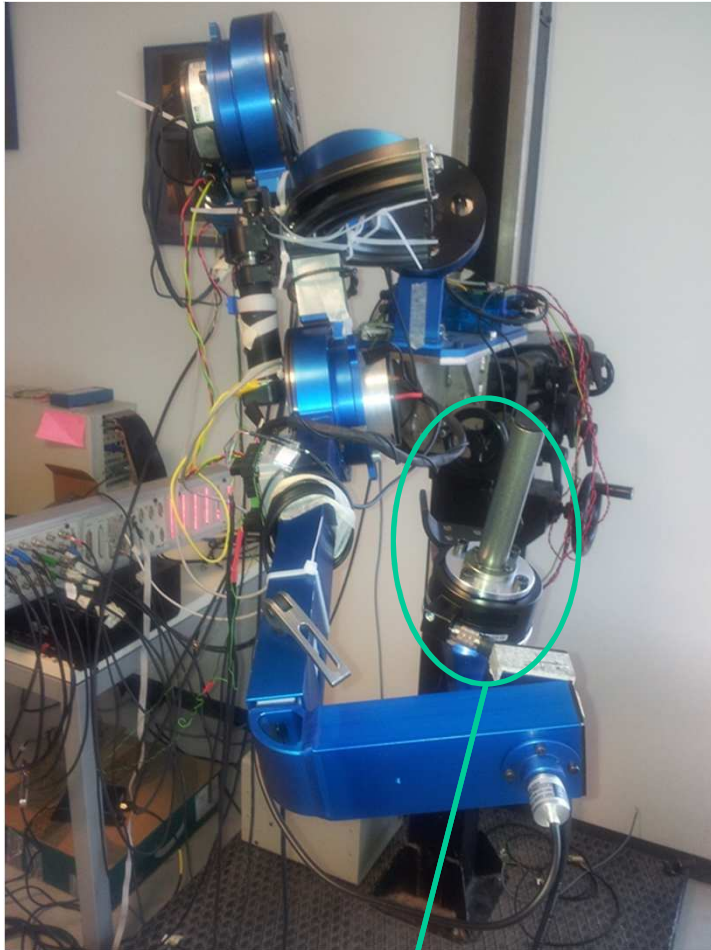
# New Exoskeleton for Upper-Limb Rehabilitation

A new exoskeleton has been developed which features:

- Isomorphism with the human arm kinematics;
- 4 actuated DOFs;
- 1 passive DOF;
- Handle with 3-axis force sensor;
- In-loco actuation;
- Modular design based on custom made robotic joints;
- Joint torque sensors.



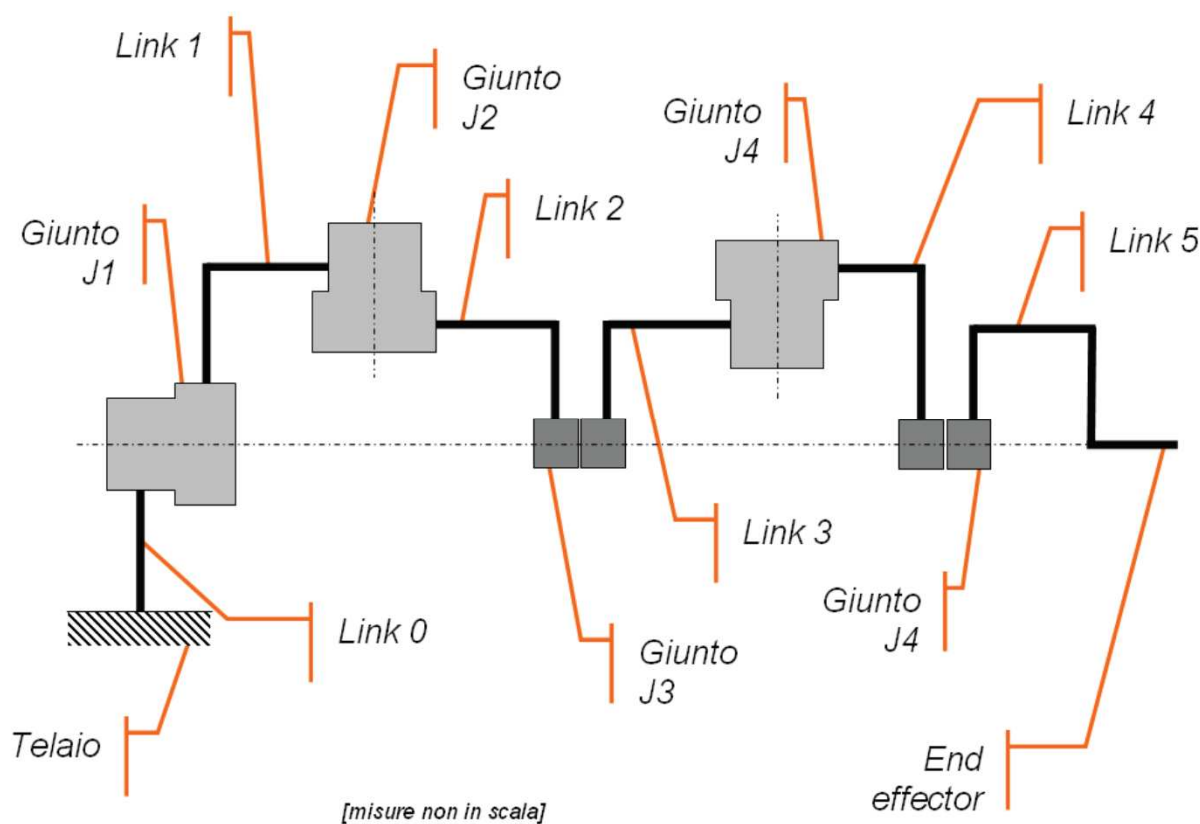
Vertechy, R.; Frisoli, A.; Dettori, A.; Solazzi, M.; Bergamasco, M.; , "Development of a new exoskeleton for upper limb rehabilitation," *Rehabilitation Robotics*, 2009. ICORR 2009. IEEE International Conference on, vol., no., pp.188-193, 23-26 June (2009)



ATI force sensor at the handle to monitor forces

# Overall architecture of the Rehab-Exos

- The design is based on 4 actuated joints and 5 dofs.



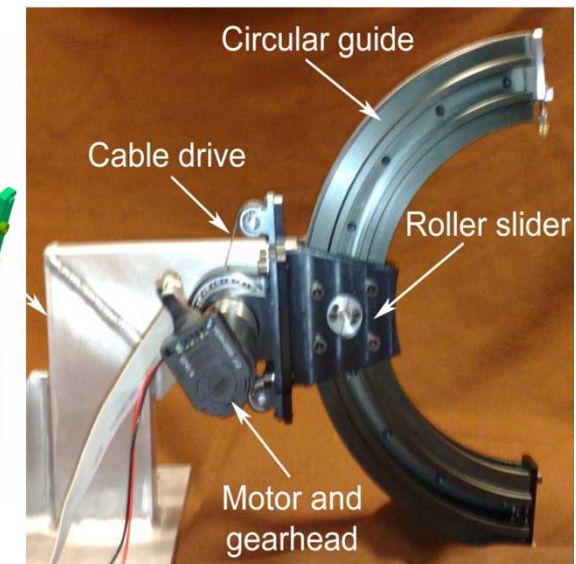
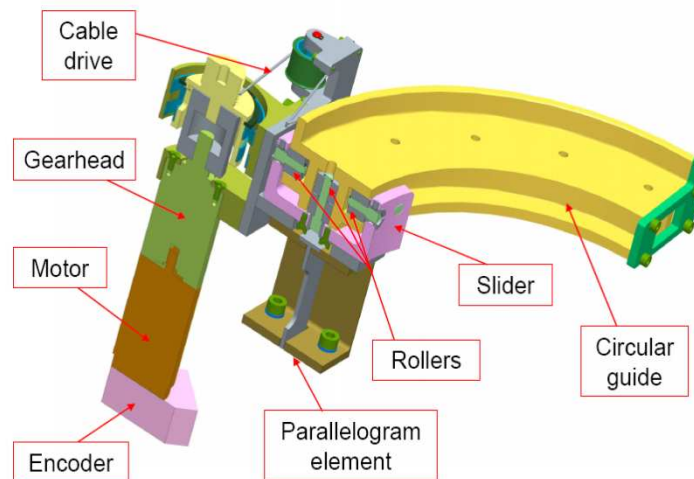
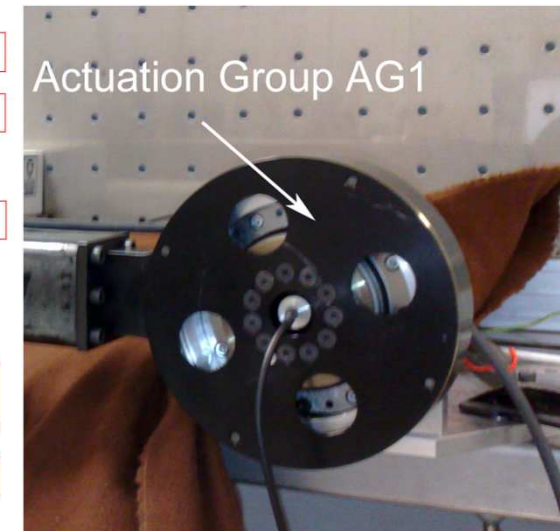
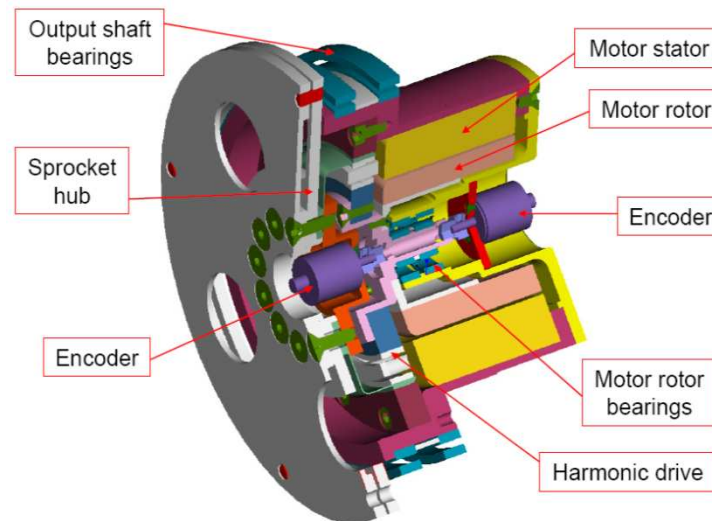
# New Robotic Joints for Rehabilitation Exoskeletons

## Novel custom made Robotic joints featuring:

- Electric motors;
- Transmissions with large reduction ratio;
- Position sensors;
- Integrated joint torque sensors;
- Inherent passive compliance.

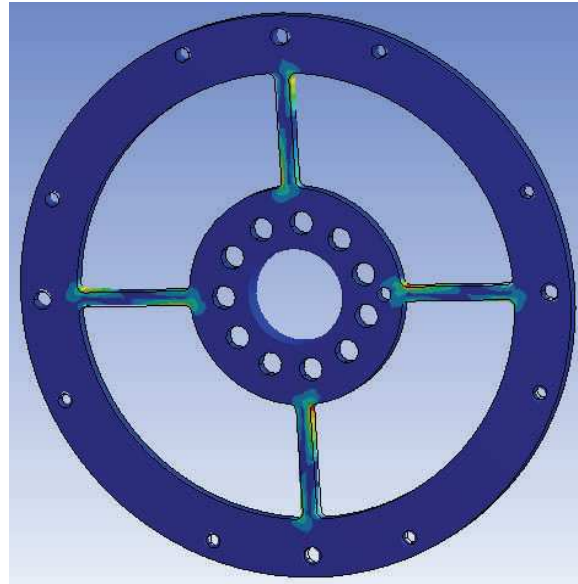
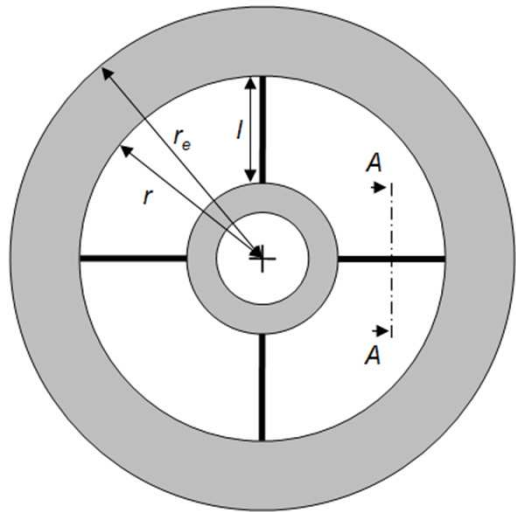
## Joint performances:

- 10rpm max. output velocity;
- 120Nm nominal maximum output torque;
- Reduced weight (3.7Kg and 2.2Kg);
- Torque sensor accuracy by 0.5% of full scale (100N·m).



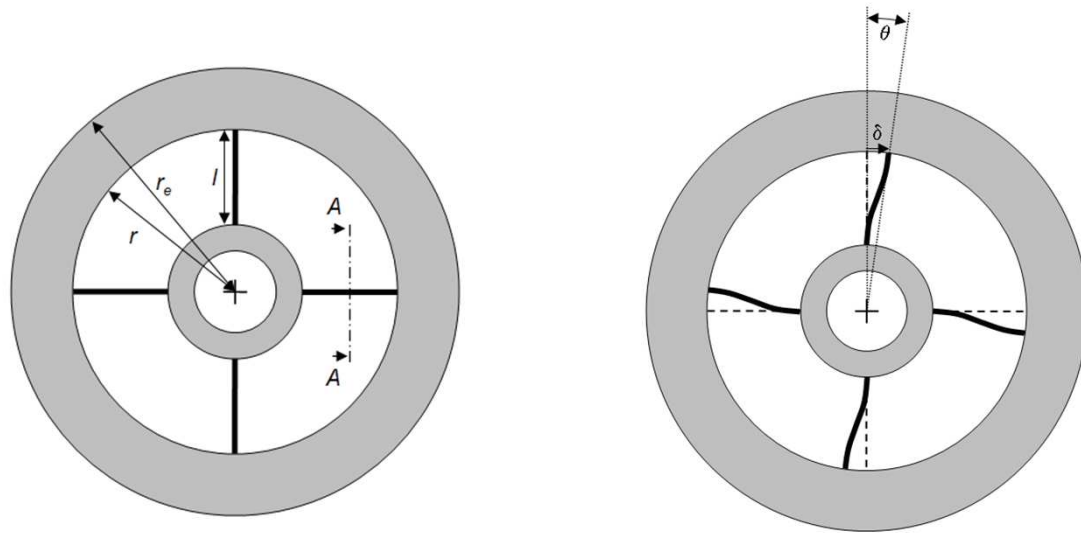


# Force sensor design

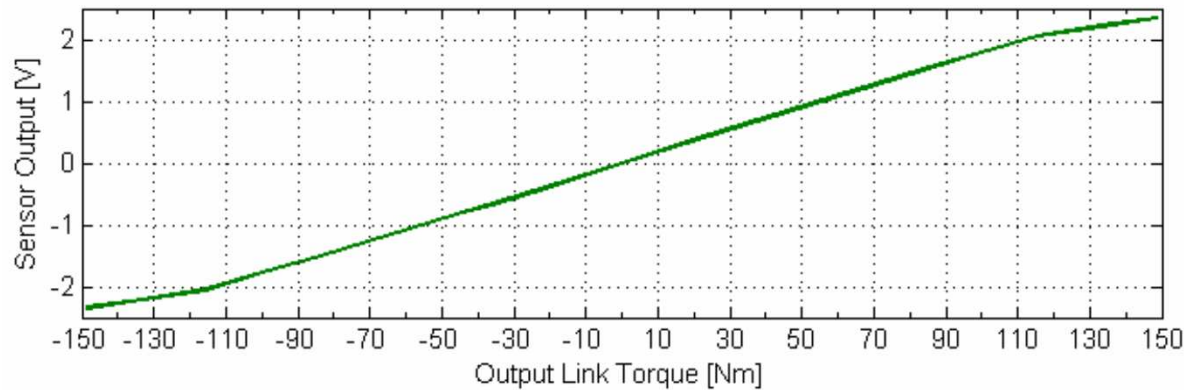


- **The employed force sensor has a cross geometry.**
- **All structure loads are supported by the joint, while only the motor torque is transmitted through the sensor.**

# Force sensor design

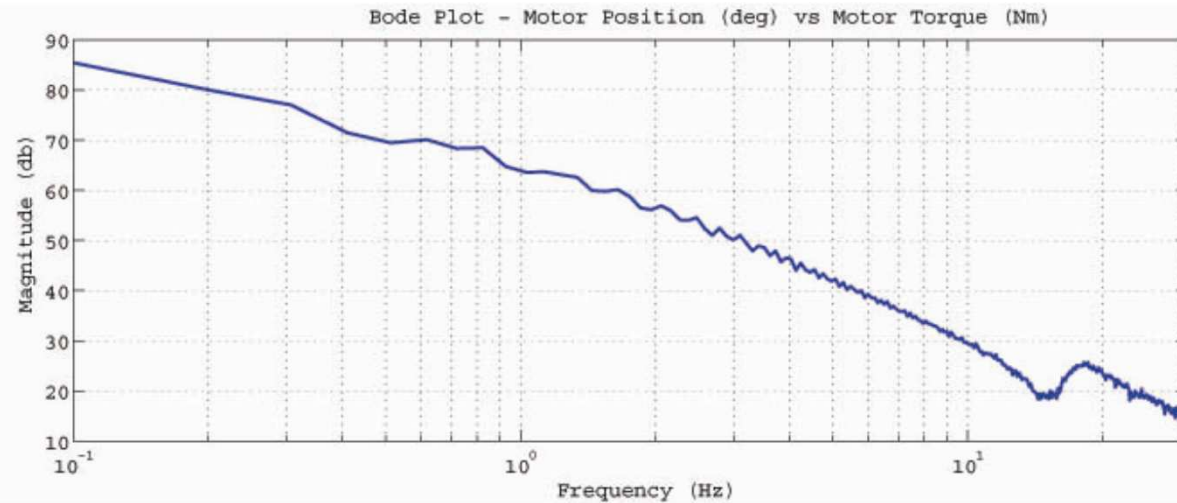


- In the operating range, the sensor presents almost a linear operating curve

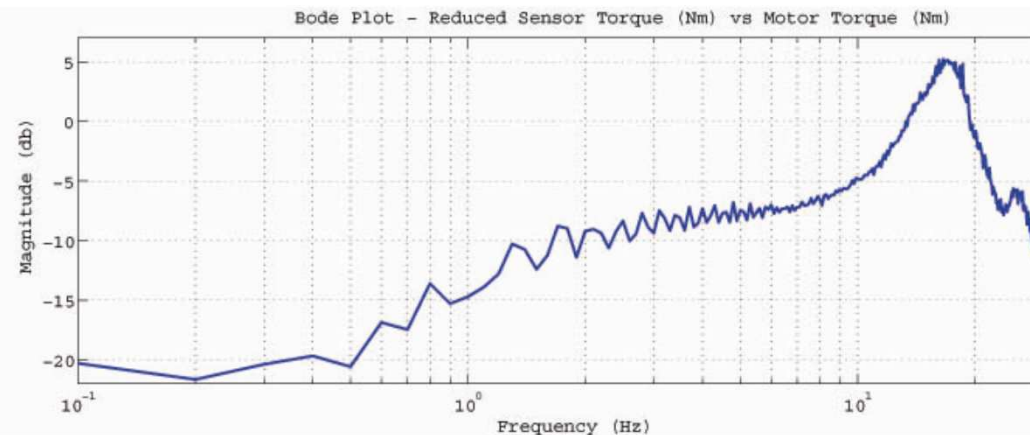


# Dynamic experimental characterization

- The system presents a dynamic bandwidth of 18Hz, as it can be seen by the experimental Bode diagrams obtained.

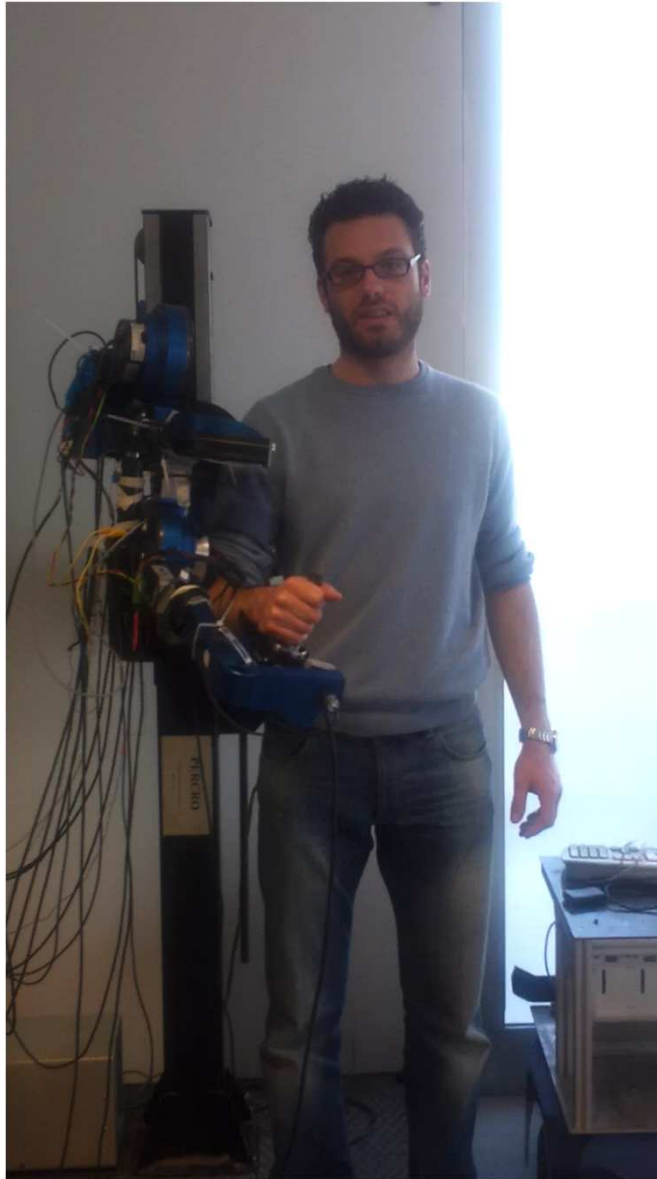


Frequency response



Force response

# System backdrivable by control

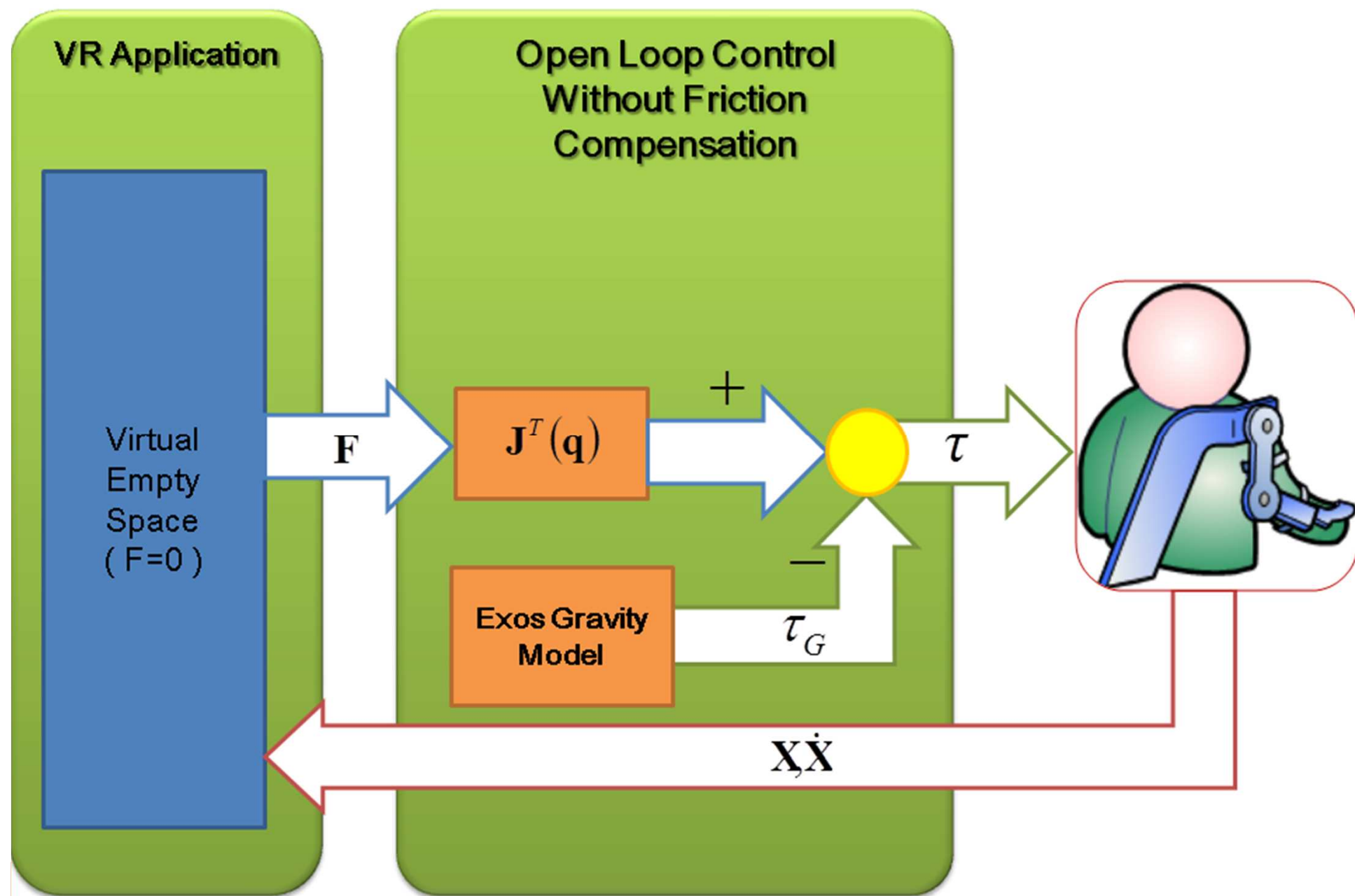


# PRINCIPLES OF CONTROL DESIGN AND PERFORMANCE EVALUATION



A tendon driven exoskeleton

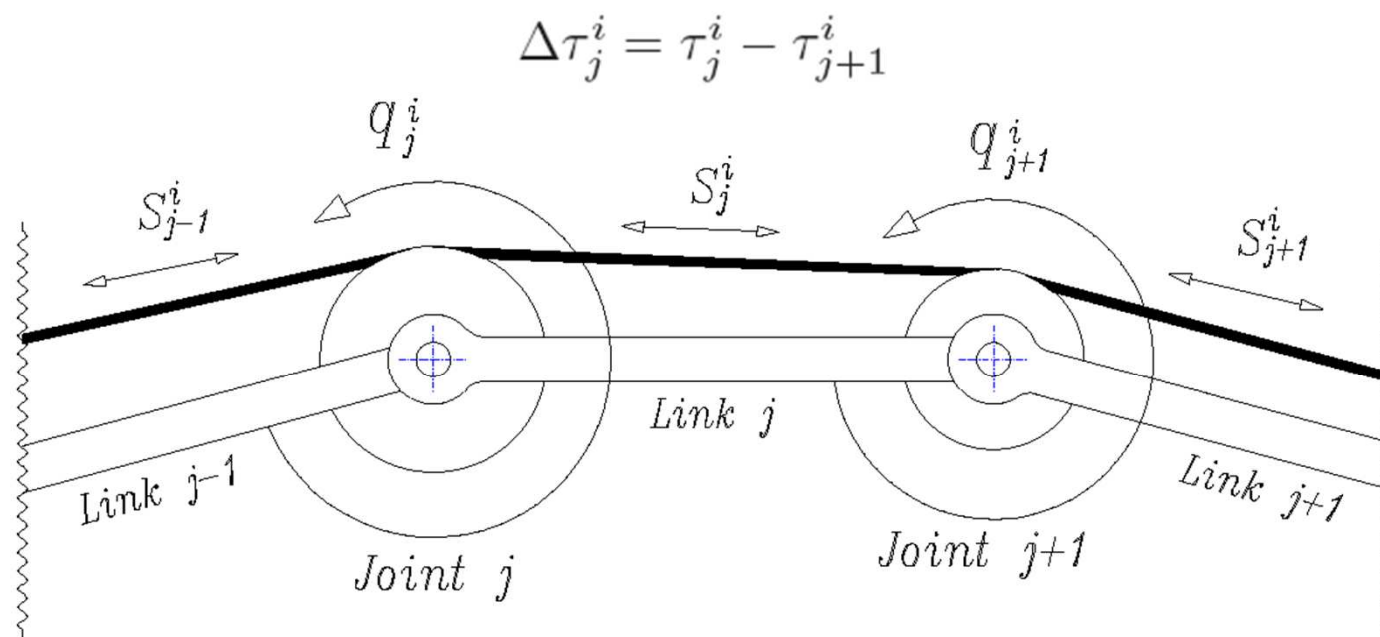
# THE L-EXOS SYSTEM



# Distributed model

A detailed model of the transmission system can be made, by dividing the cable in branches, each branch connecting two joints.

## TENDON TENSION LOSS



Simone Marcheschi, Antonio Frisoli, Carlo Alberto Avizzano, Massimo Bergamasco , "A Method for Modeling and Controlling Complex Tendon Transmissions in Haptic Interfaces ", Proceedings of IEEE ICRA 2005, April 2005, Barcelona



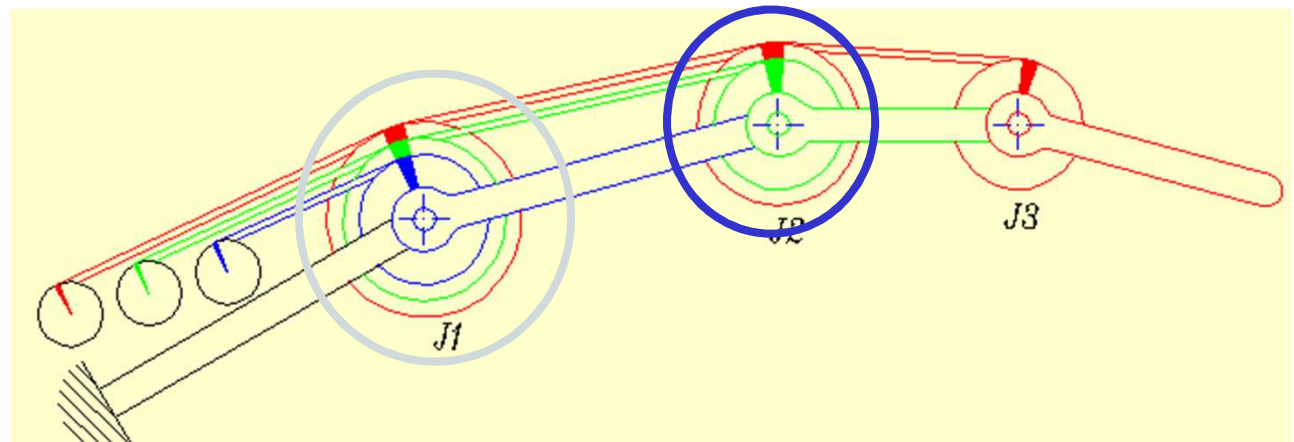
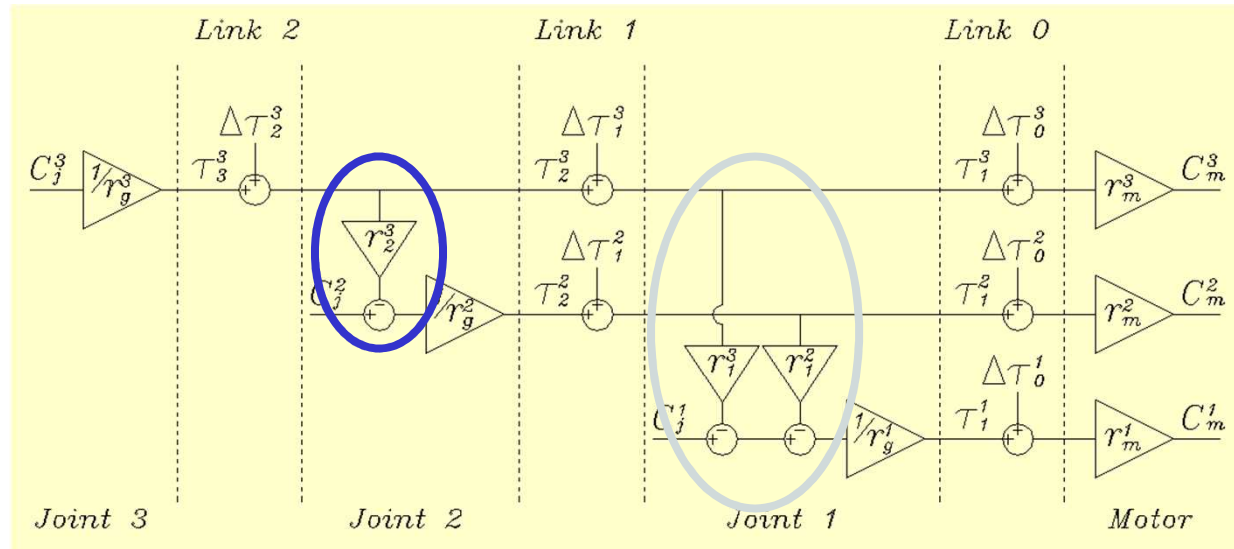
# Static model

**Torque transmission along a multi-joint transmission system.**

**If the cable transmission is not ideal, a tension loss will happen due to dissipative phenomena.**

**If we consider the transmission  $i$ , the loss of tension between joint  $j$  and joint  $j+1$  is:**

$$\Delta\tau_j^i = \tau_j^i - \tau_{j+1}^i$$



# Dissipative effects

The most important dissipative phenomena which affect cable transmissions are the following:

- friction on pulley supports;
- efficiency of cable transmissions;
- efficiency of speed-reducers.

$$M = \begin{cases} M_{ext} & \text{if } n = 0 \text{ and } M_{ext} < \alpha p \\ \beta p + h(n) & \text{if } n \neq 0 \end{cases}$$

$$h(n) = \begin{cases} h_0 & \text{if } n < n_0 \\ h_1 n^{2/3} & \text{if } n \geq n_0 \end{cases}$$

## Efficiency of cable transmissions

The efficiency of cable transmission is related to following phenomena:

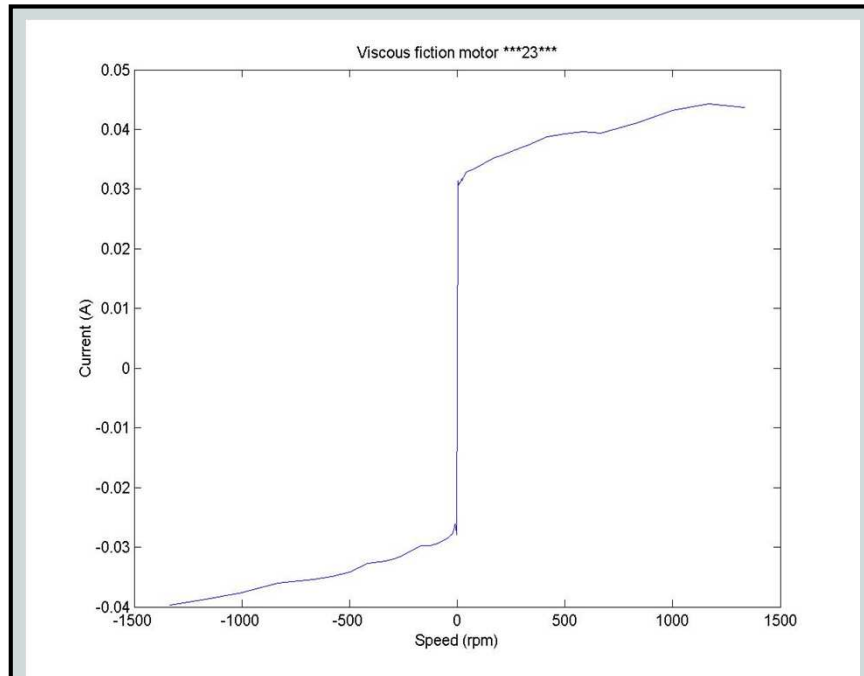
- elastic sliding between cable and driven pulley;
- internal friction between strands which compose the cable;
- friction induced by misalignment between cable and pulley.

Also the efficiency of the reducer should be considered. Direct and inverse reducer efficiency should be considered according to the actual direction of motion.

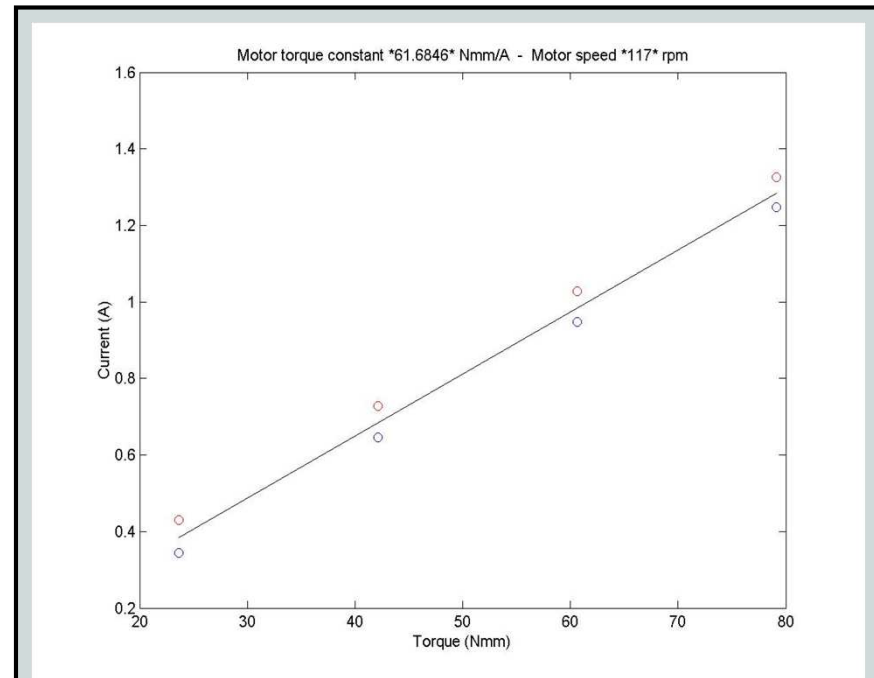
$$M_{out} = \begin{cases} \eta_d \kappa M_{in} & \text{if } M_{out} n_{rid} \geq 0 \\ \frac{\kappa M_{in}}{\eta_i} & \text{if } M_{out} n_{rid} < 0 \end{cases}$$

$$\eta_i = \frac{2\eta_d - 1}{\eta_d}$$

# Motor characterization of friction



**By recording the current for moving the motor at different speeds with no load, it's possible to define how friction depends on shaft speed**



**By recording the current for moving the motor with different loads, and at fixed speed, it's possible to define how friction depends on load.**

# Hypotheses and simplified model

## Base operative hypotheses:

- **The efficiency of reducers and cable transmissions can be neglected with respect to the bearings friction**
- **Static friction phenomena are not characterized**
- **Viscous friction can be expressed as a linear function of bearing speed**
- **All contributions due to the dynamic friction torques of all bearings of a given transmission and placed on a given link can be represented as a single equivalent torque**

**All dissipative phenomena can be represented as a loss in the cable tension which goes through a given link j by this relation.**

**It's a linear function of cable tension and cable speed**

$$\Delta \tau_j^i = \beta_j^i \tau_j^i + \nu_j^i \dot{S}_j^i$$

**Further approximation: the friction forces are considered independent of the cable tension.**

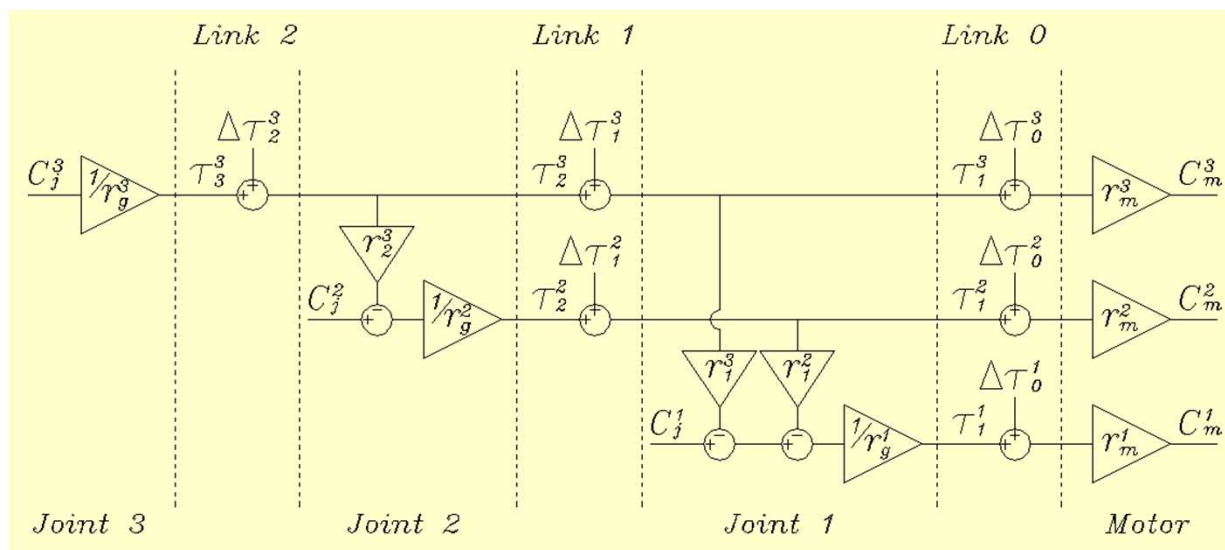
**A term to consider the static friction due to the tension of the transmission is also added.**

$$\Delta \tau_j^i = \tau_{0,j}^i + \nu_j^i \dot{S}_j^i$$

# Algorithms for feedforward estimation of compensation torques

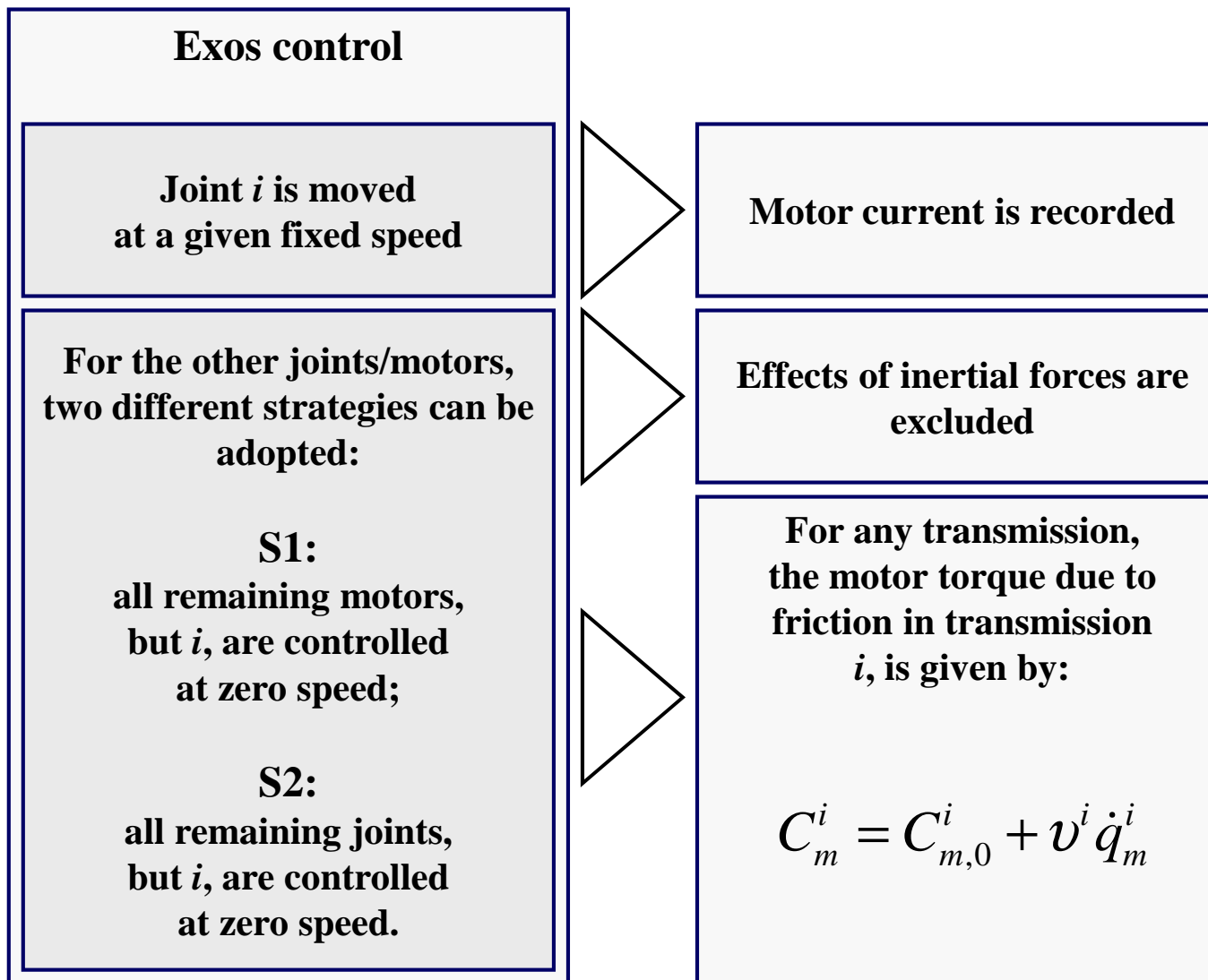
Compensation torques can be found, starting by the desired value of joint torques, and can be computed according to

$$\Delta \tau_j^i = \tau_{0,j}^i + v_j^i \dot{S}_j^i$$



**Identification of unknown parameters required**

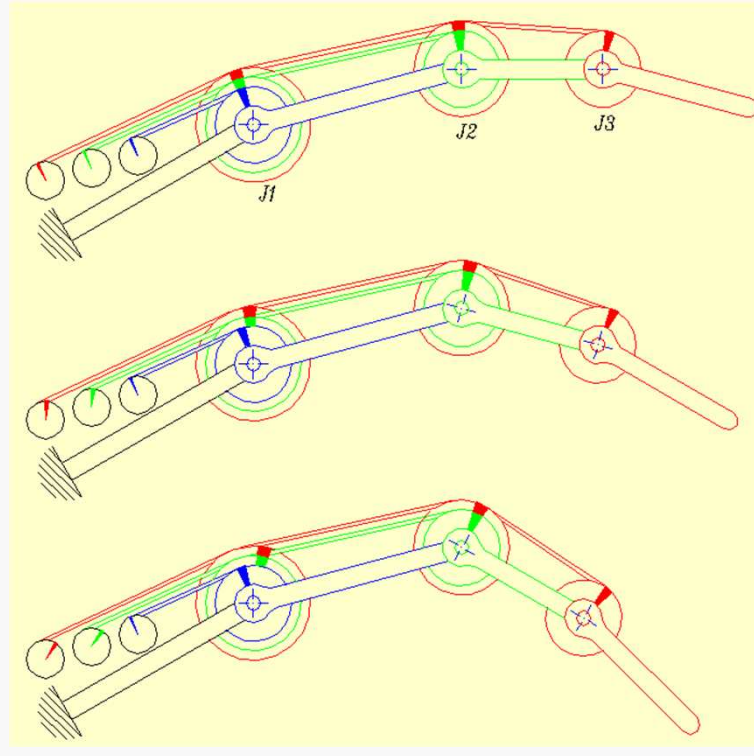
# Experimental identification procedure



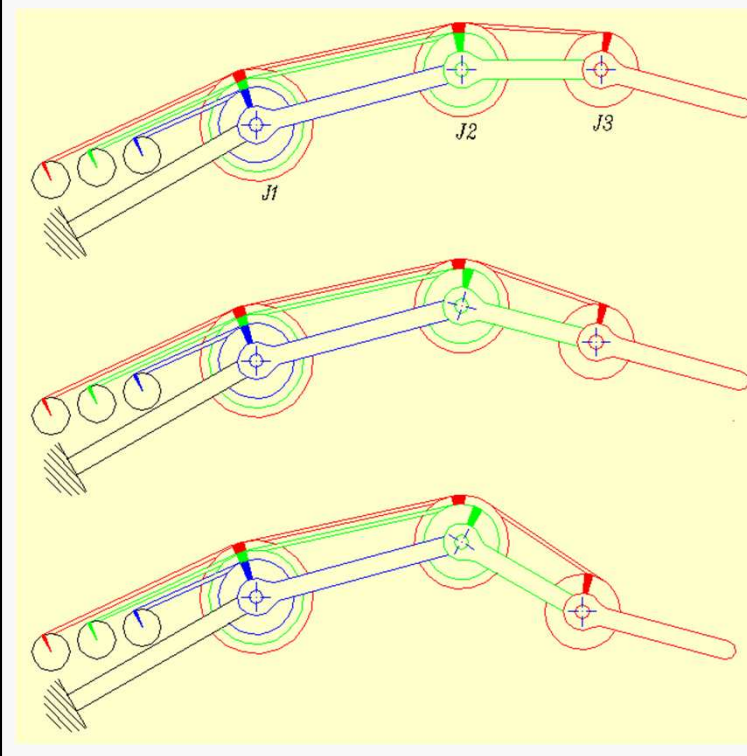
# Kinematics of coupled transmissions

Joint  $i$  is moved: transmissions of joint  $k$ , with  $k \geq i$ , will present a movement

**Blocked joints**



**Blocked motors**



**S2. all remaining joints, but  $i$ , are controlled at zero speed.**

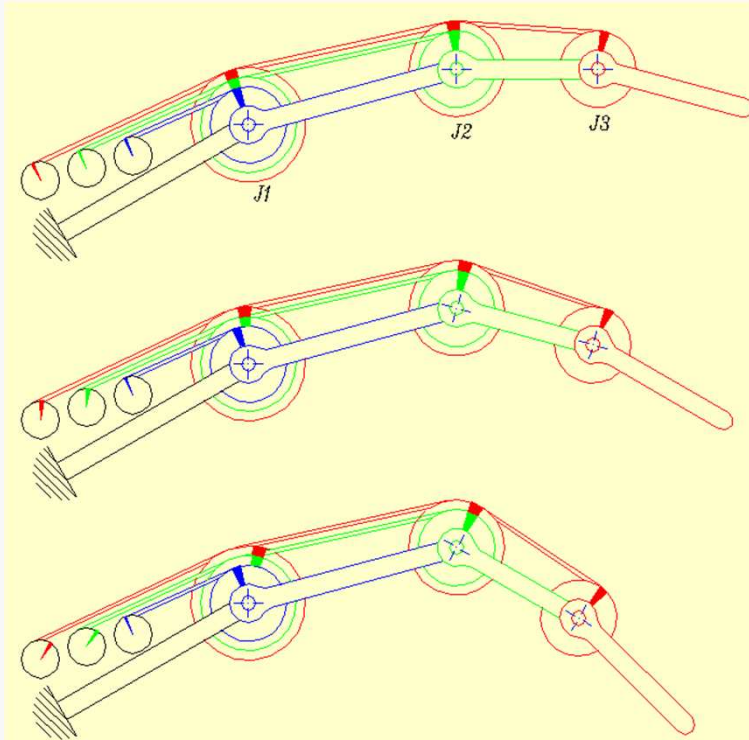
**S1. all remaining motors, but  $i$ , are controlled at zero speed;**



# Condition S2

Joint  $i$  is moved: transmissions of joint  $k$ , with  $k \geq i$ , will present a movement

## Blocked joints



For  $i < j$

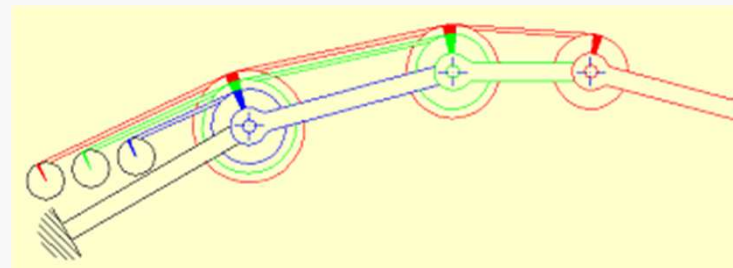
$$\dot{q}_{mi} = 0$$

For  $i \geq j$

$$\dot{q}_{mi} = k_{ij} \dot{q}_j$$

$$\dot{\mathbf{q}} = \begin{bmatrix} 0 \\ \vdots \\ \dot{q}_i \\ \vdots \\ 0 \\ 0 \end{bmatrix}$$

$$\dot{\mathbf{q}}_m = \mathbf{K}_T \dot{\mathbf{q}} = \begin{bmatrix} k_{11} & 0 & 0 & \dots & 0 \\ k_{21} & k_{22} & 0 & \dots & 0 \\ k_{31} & k_{32} & k_{33} & \dots & 0 \\ \dots & \dots & \dots & \dots & 0 \\ k_{n1} & k_{n2} & k_{n3} & \dots & k_{nn} \end{bmatrix} \dot{\mathbf{q}}$$



08 March 2012

# Energetic balance of losses

If we assume that at the motor side:

$$\tau^i = \tau_0^i + \nu^i \dot{q}_m^i$$

Tendon loss at each joint:

$$\Delta \tau_j^i = \tau_{0,j}^i + \nu_j^i \dot{S}_j^i$$

Estimation of power dissipated:

$$\dot{S}_j^i = k_j'^i \dot{q}_m^i$$

$$E_{loss,i} = \tau_0^i \dot{q}_m^i + \nu^i \dot{q}_m^i{}^2 = \dot{q}_m^i \sum_j \tau_{0,j}^i k_j'^i + \dot{q}_m^i{}^2 \sum_j \nu_j^i k_j'^i{}^2$$

This implies:

$$\nu^i = \sum_j \nu_j^i k_j'^i{}^2$$

↑
↑

**Known**
**Unknown**

$$\tau_0^i = \sum_j \tau_{0,j}^i k_j'^i$$

↑
↑

**Known**
**Unknown**

# Energetic balance of losses

If we use the following model

$$\hat{\nu}_j^i = \gamma_j^i \nu^i \quad \nu^i = \sum_j \nu_j^i k_j'^{i2}$$

Known
Unknown

$$\tau_0^i = \sum_j \tau_{0,j}^i k_j'^{i2} \quad \hat{\tau}_{0,j}^i = \alpha_j^i \tau_0^i$$

Known
Unknown

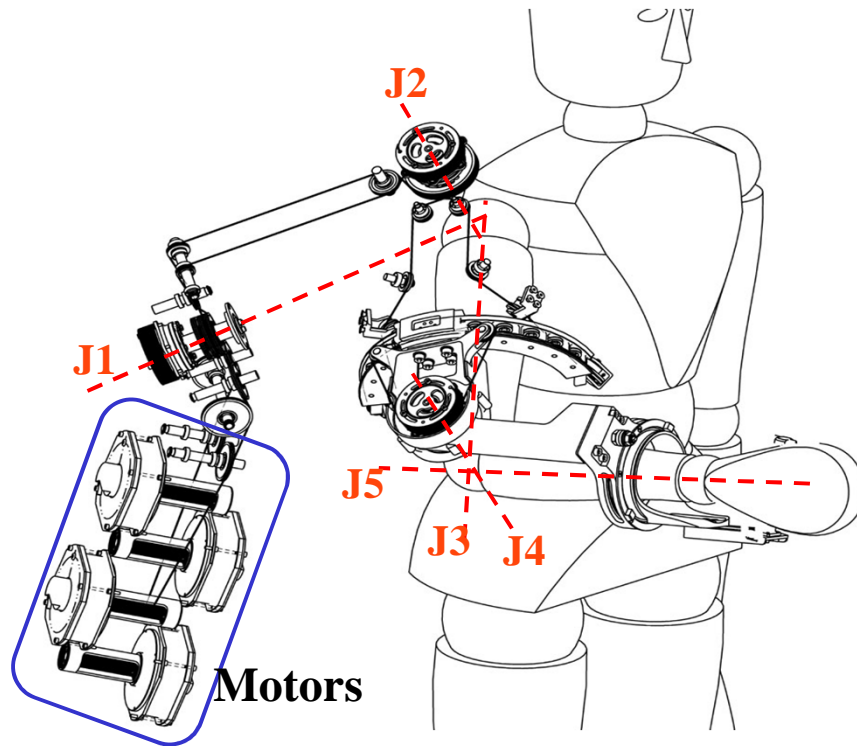
  

$$1 = \sum_j \gamma_j^i k_j'^{i2} = \sum_j p_j^i$$
  

$$1 = \sum_j \alpha_j^i k_j'^{i2} = \sum_j p_j'^i$$

**Physical interpretation:** the terms  $p_j^i$  and  $p_j'^i$  represent the percentage losses that we have along the transmission due to friction.

# Exos transmission system



$$\Delta \tau_j^i = \tau_{0,j}^i + \nu_j^i \dot{S}_j^i$$

$$1 = \sum_j \gamma_j^i k_j^{i2} = \sum_j p_j^i \quad 1 = \sum_j \alpha_j^i k_j^i = \sum_j p_j^i$$

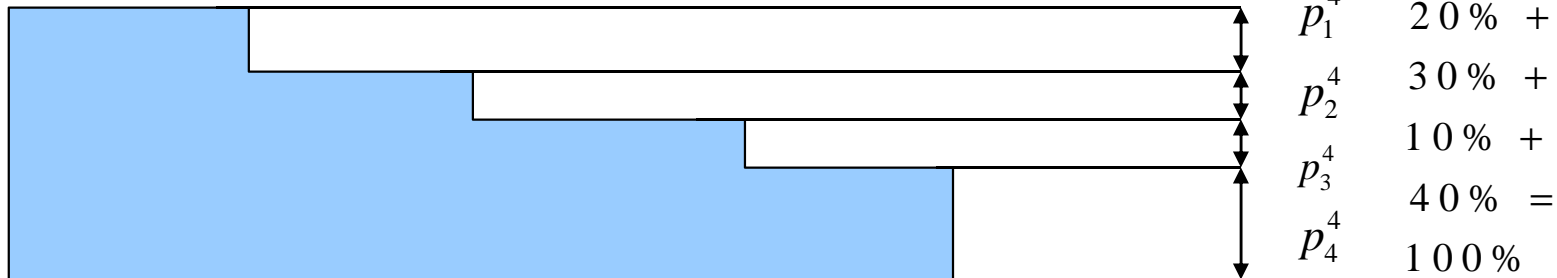
Experimental identification of:

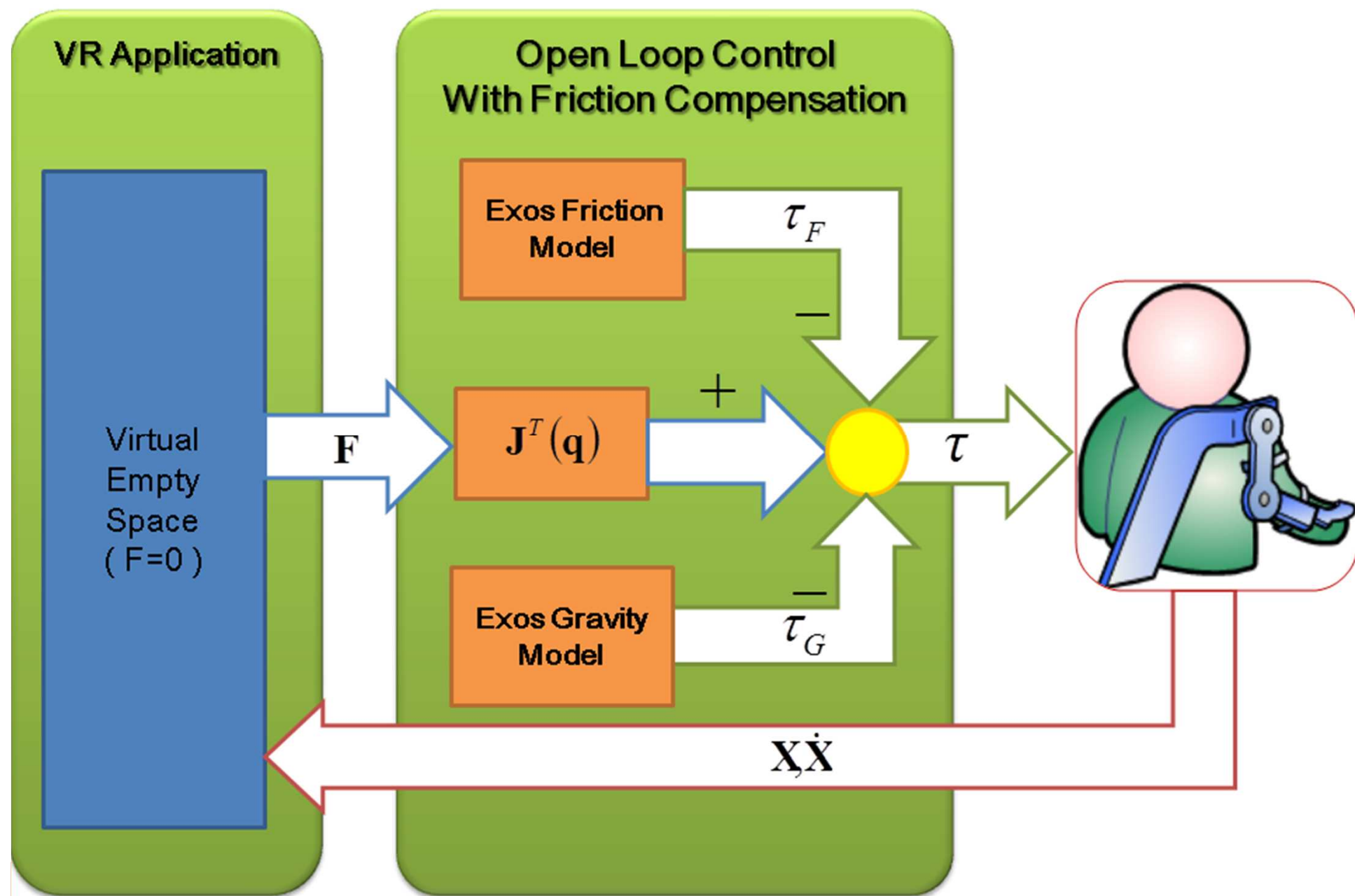
$$\tau^i = \tau_0^i + \nu^i \dot{q}_m^i$$

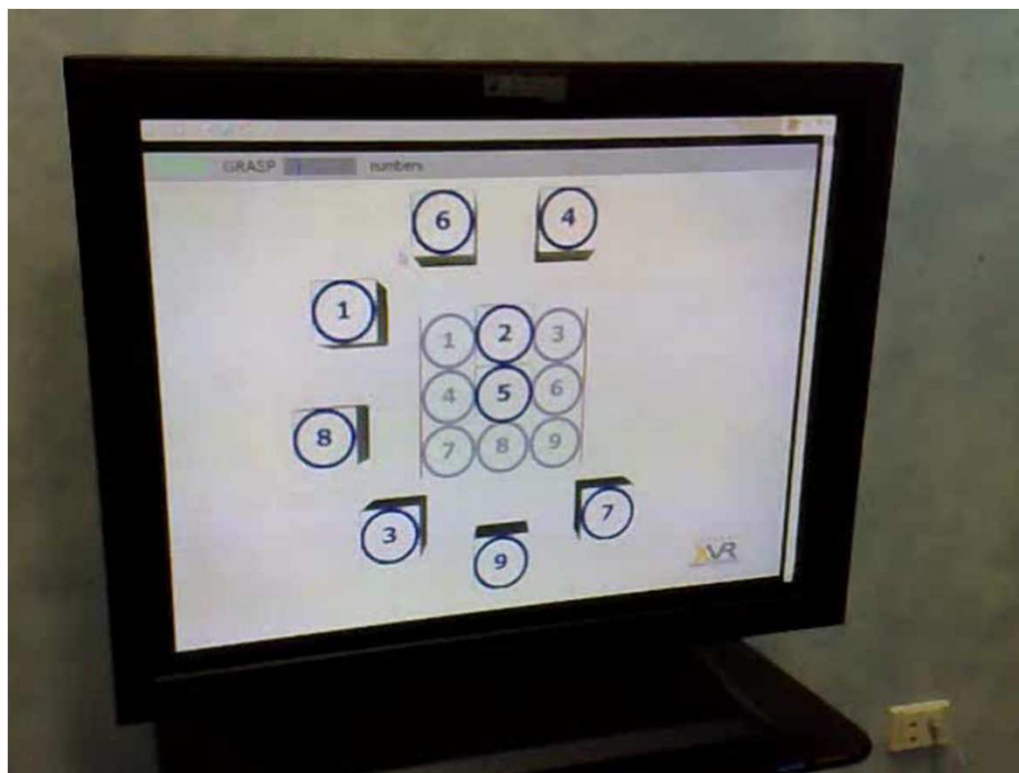
Estimation of percentage losses by design

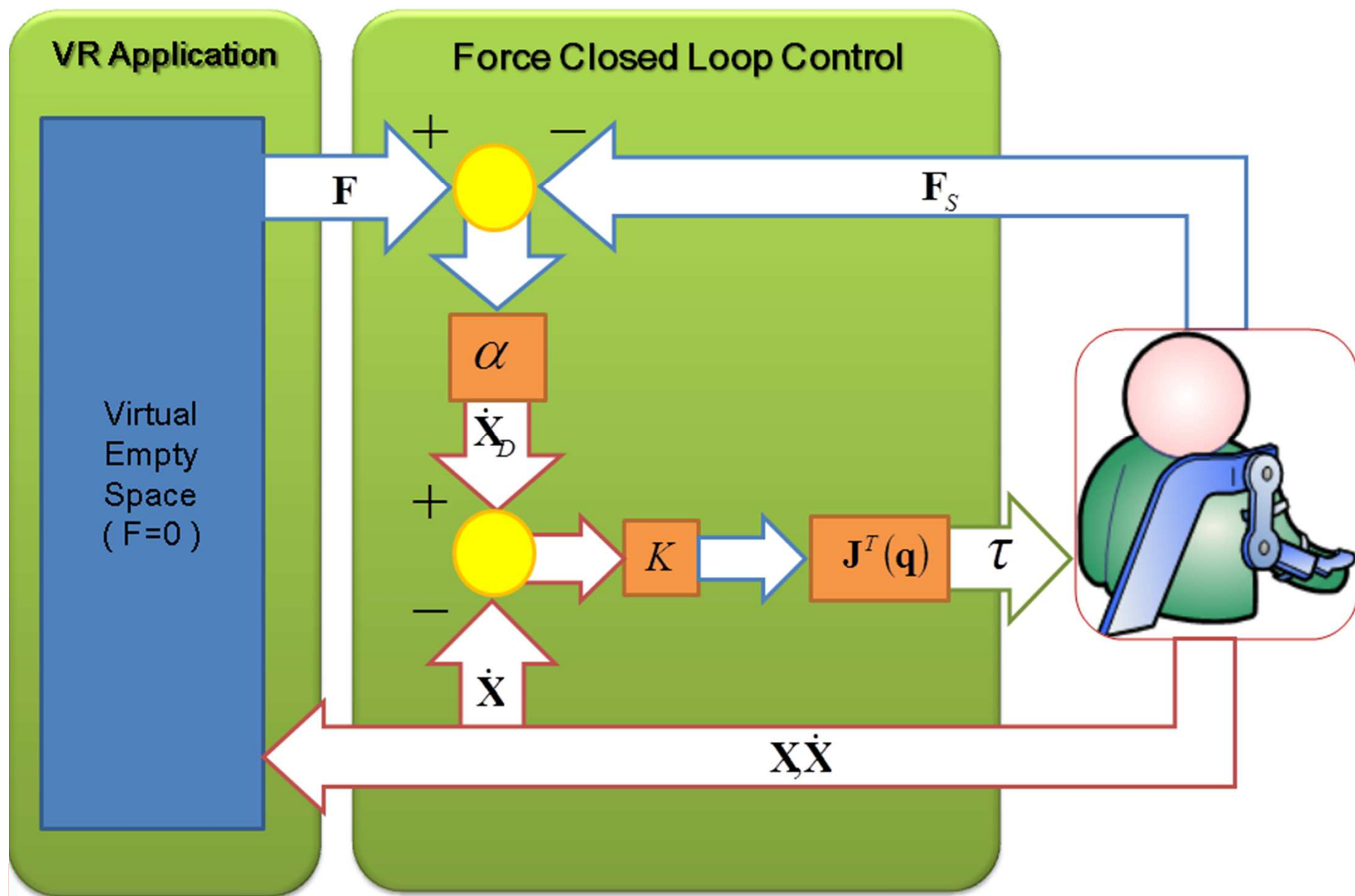
$$\hat{\nu}_j^i = \gamma_j^i \nu^i \quad \hat{\tau}_{0,j}^i = \alpha_j^i \tau_0^i$$

Tension loss in percentage

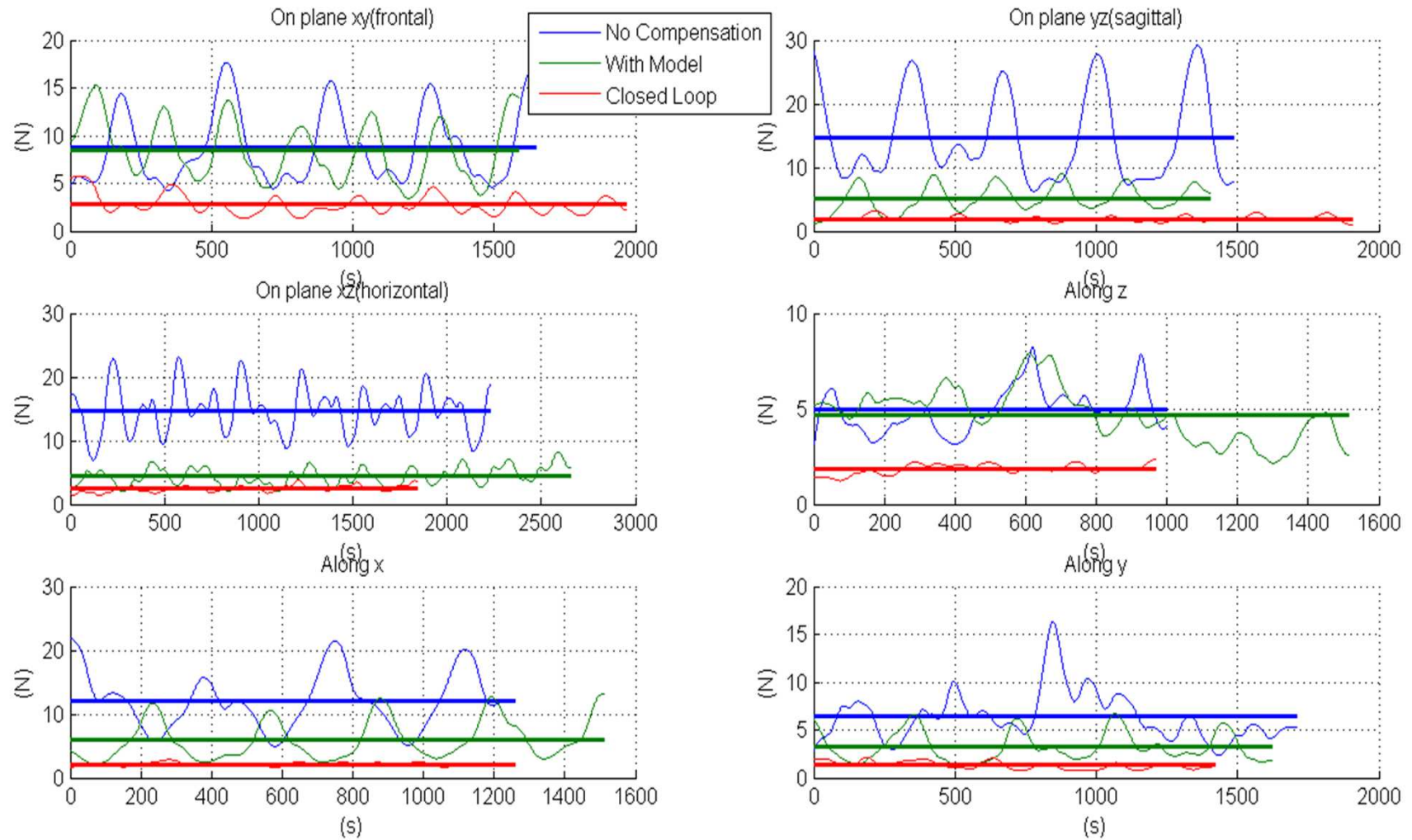






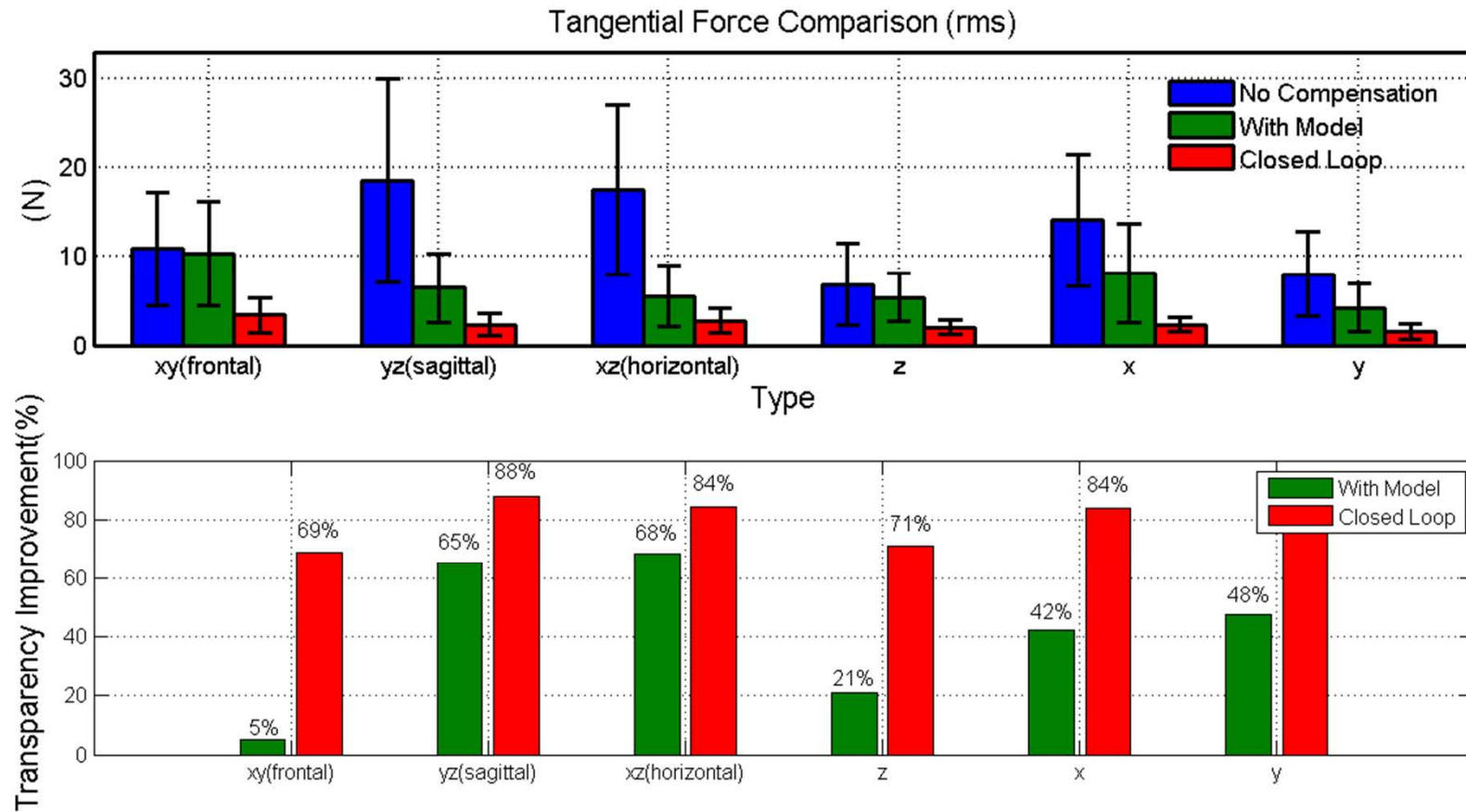


# Force performance of the L-Exos system





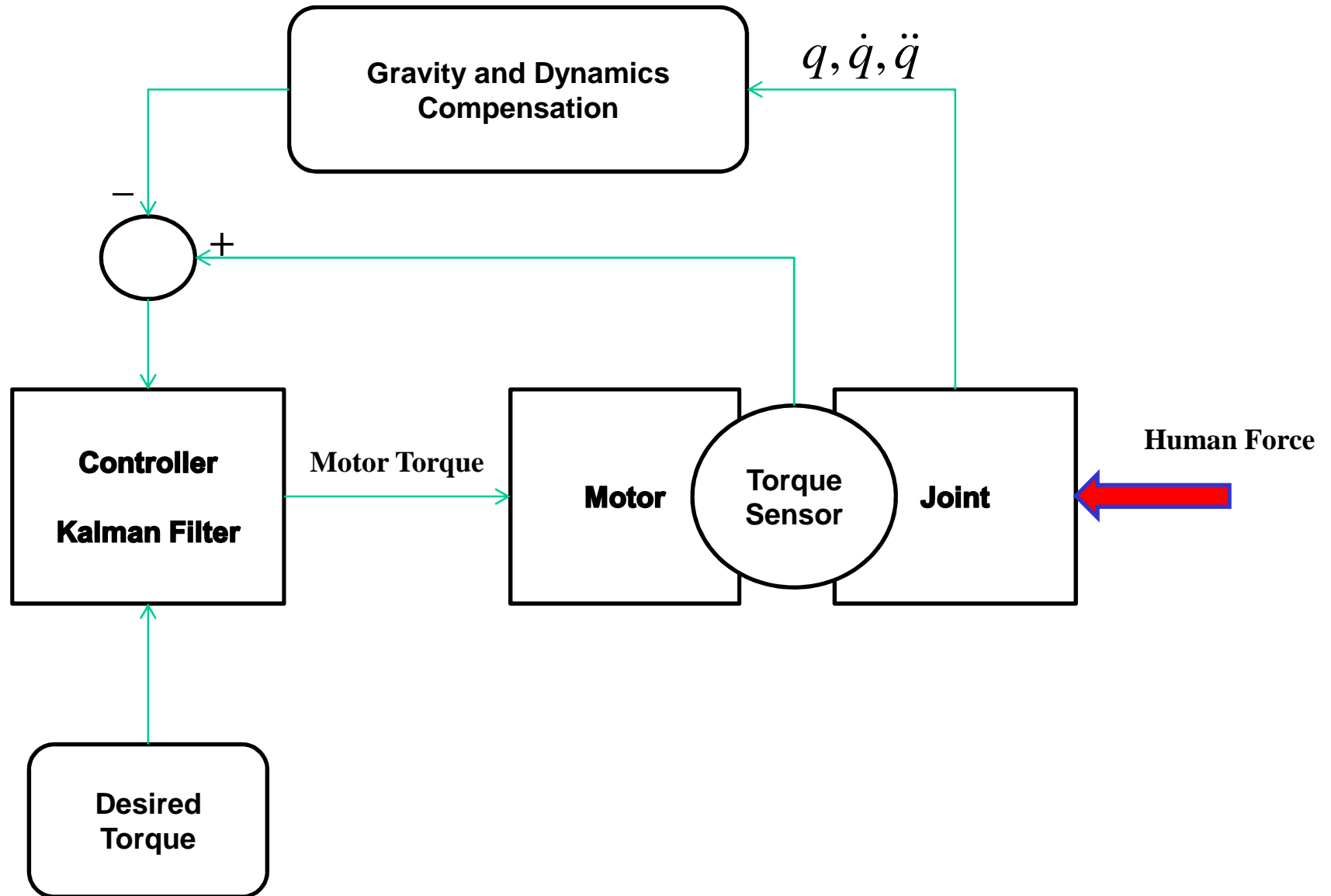
# Error in force in different conditions

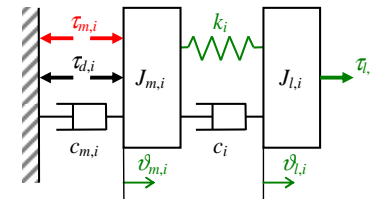
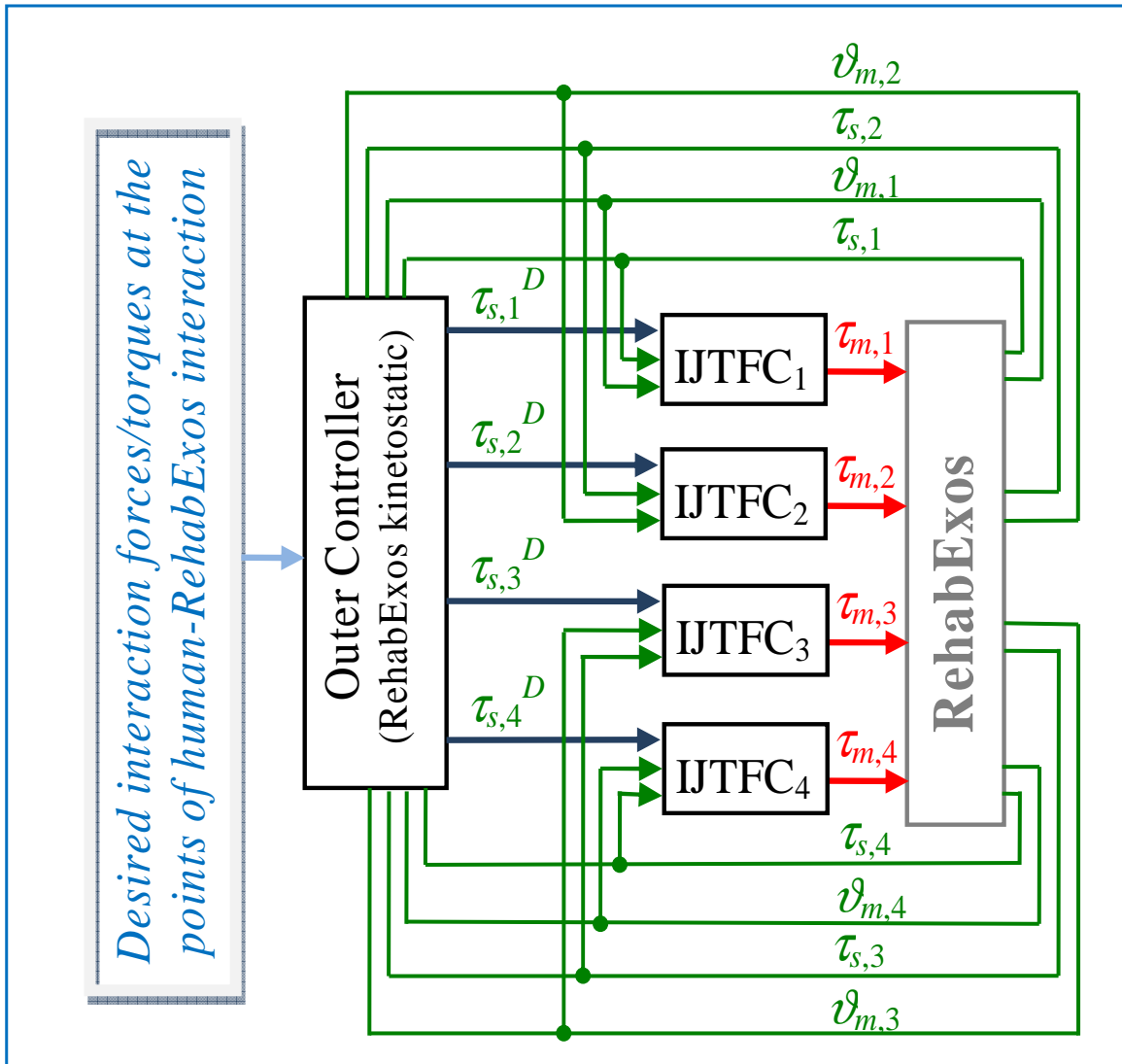


# THE REHAB-EXOS SYSTEM



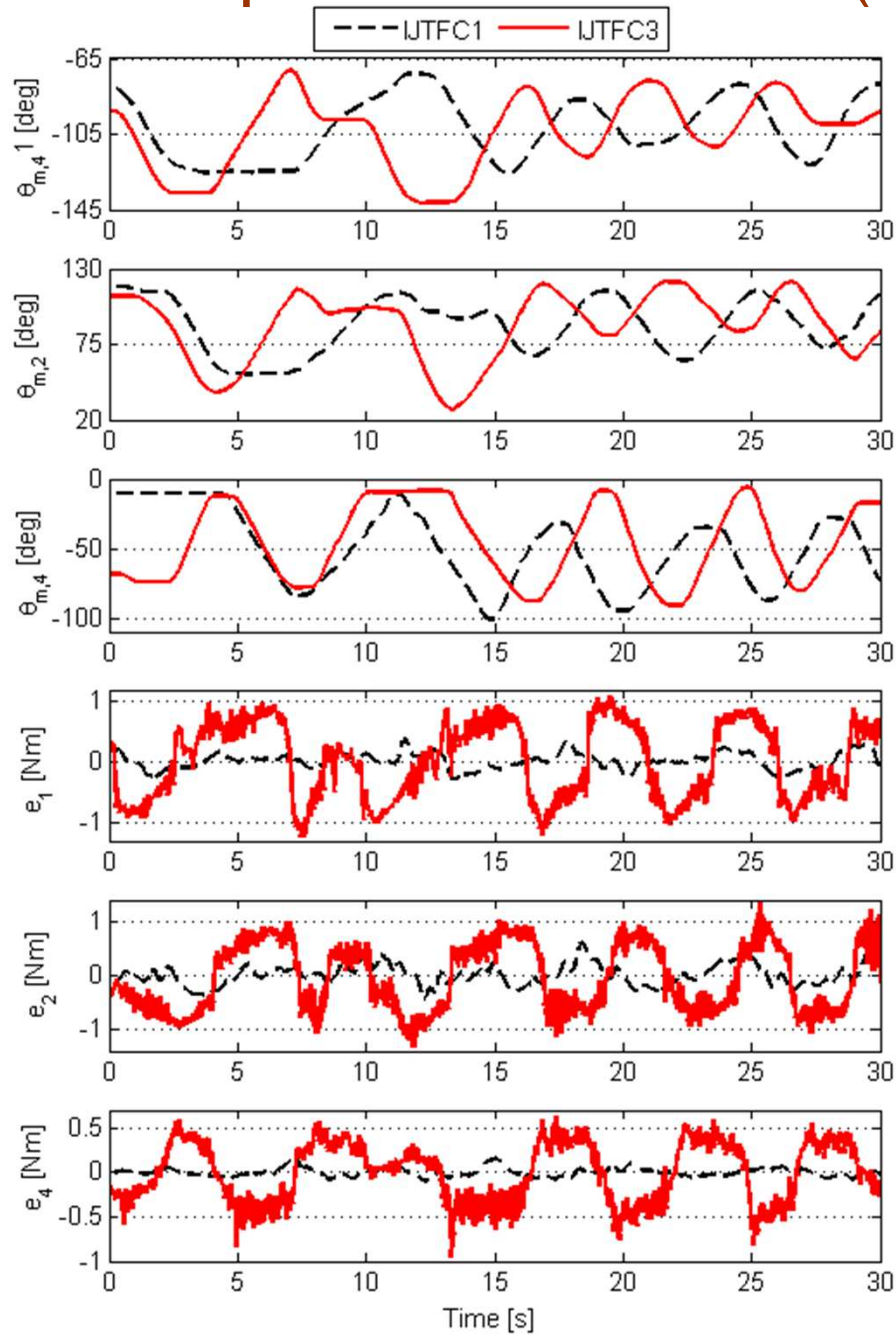
# General control scheme





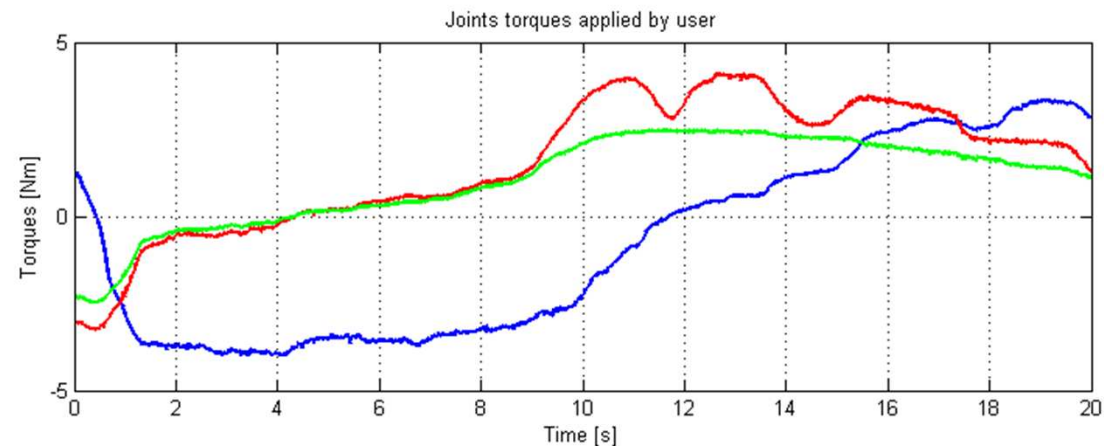
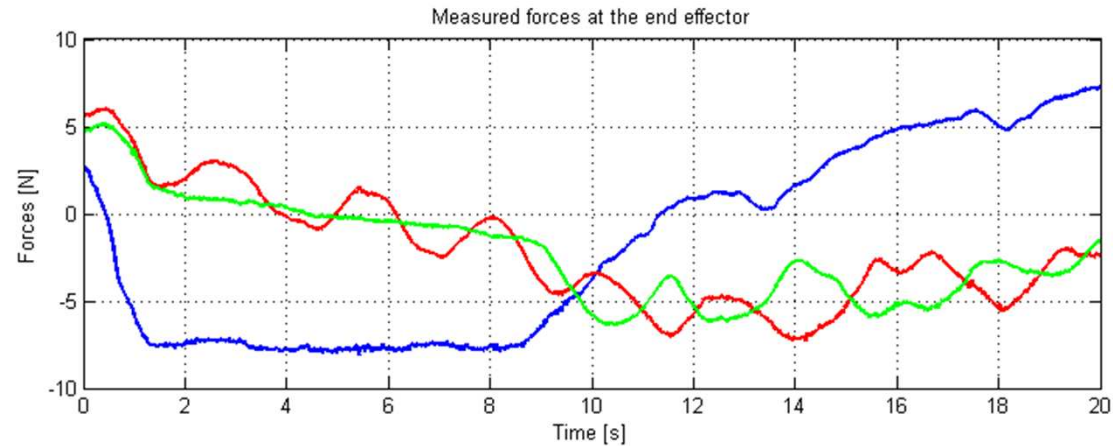
Vertechy, R., Frisoli A, Solazzi M., Dettori A., Bergamasco M., Linear-Quadratic-Gaussian Torque Control Application to a Flexible Joint of a Rehabilitation Exoskeleton, IEEE ICRA International Conference on Robotics and Automation, Anchorage, Alaska, 2010

# Experimental values (1/2)



**Comparison of  
IJFC1 state space model of  
dynamics  
IJFC3 Standard PID force control**

# Experimental values (2/2)



**Current limitations of this evaluation are:**

- **Only 50% dynamic compensation provided for joint #1**
- **20% feedforward compensation for joint #2**
- **0% for joint #3**
- **Inperfect calibration of joint torque sensor**

# Conclusions

- In exoskeleton design both low impedance and high impedance design might be efficiently used

## Impedance designs

### PROs

- Backdrivable mechanics
- Delocalization of actuation

### CONTRAs

- Joint coupling by tendon transmissions
- Difficulty in compensation and estimation of static friction effects
- Dynamic compensation

## Admittance designs

### PROs

- System safety is motors are turned off
- Precise control of dynamics/friction
- Joint torque monitoring

### CONTRAs

- Need of force sensors in the structure
- Possible limitation in maximum speed in high reduction designs

thank you!

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